

# Assessment of design models for aluminium alloy CHS beam-columns with stocky cross-sections

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## **ABSTRACT**

There is little information on the prediction accuracy of the international design standards for aluminium alloy circular hollow section (CHS) beam-columns. To bridge this gap, this paper presents a comprehensive numerical investigation on the performance of aluminium alloy CHS beam-columns with stocky cross-sections. A non-linear finite element (FE) model was developed and validated against experimental results. The validated FE model was employed to conduct an extensive parametric study of aluminium CHS members subjected to combined axial compression and bending over a range of lengths, cross-sections and applied eccentricities. The effect of these parameters was investigated for two material grades, 6061-T6 and 6082-T6, resulting in a total of 168 FE models. The numerically obtained capacities were used to assess the accuracy of the strength predictions of Eurocode 9 (EC9), the Aluminium Design Manual (AA) and the Direct Strength Method (DSM). On the basis of the results, EC9, with the exponent of the interaction formula,  $\psi_c$ , calculated as a function of the buckling reduction factor  $\chi$ , was found to provide the most accurate strength predictions with a mean value equal to 1.01 for both examined material grades. The most conservative predictions were given by the DSM, whilst the most scattered predictions by the AA, with a Coefficient of Variation (COV) equal to 0.17. Finally, based on the DSM buckling strength

32 and the plastic cross-sectional moment resistance, a new interaction curve is proposed for CHS  
33 beam-columns with stocky cross-sections.

34

35 **Keywords:** Aluminium Alloy; Circular Hollow Sections; Beam-columns; Eurocodes; Direct  
36 Strength Method; Aluminum Association

37

38

## 39 1. Introduction

40 Hollow aluminium structural members are extensively utilised within the aerospace and  
41 automotive sectors, owing to the combined high strength-to weight ratio and flexibility. On the  
42 contrary, the adoption of aluminium in the construction industry is still limited. Due to  
43 aluminium's Young's Modulus being roughly one-third that of steel, the design of structural  
44 members poses a challenge, as they are prone to instability induced failure modes and excessive  
45 deflections.

46 In recent years, there has been a significant increase in research studies investigating  
47 different cross-sections and members made from structural aluminium alloys (Georgantzia et  
48 al., 2021a). Even though considerable emphasis has been placed on square and rectangular  
49 hollow sections (SHS, RHS) (e.g. Su et al., 2016; Wang et al., 2016; Castaldo et al., 2017; Feng  
50 et al., 2019; Georgantzia et al., 2021b; Bock et al., 2021; Hu et al., 2021; Li et al., 2022a; Li et  
51 al., 2022b; Li et al., 2022d; Rong et al., 2022a; Ali et al., 2022; Zhi et al., 2022; Nastri et al.,  
52 2022), studies on the performance of aluminium alloy circular hollow sections (CHS) remain  
53 relatively limited. In particular, the performance of circular hollow sections was studied by  
54 Wang et al. (2015, 2016) who examined experimentally and numerically the performance of  
55 aluminium alloy 6082 circular tubes in axial compression. Feng et al. (2016, 2020) focused on  
56 perforated CHS aluminium beams and columns. Zhu and Young (2008) examined the  
57 performance of CHS columns, whilst recently Li et al. (2022c) investigated the ultimate  
58 response of CHS 7A04-T6 columns. All these studies focus either on axial compression or pure  
59 bending performance without any consideration of the combined compression and bending  
60 response.

61 Aluminium circular hollow sections are highly vulnerable to buckling failures when  
62 subjected to a combination of axial compression and bending, therefore their response under  
63 this loading regime requires careful consideration. The authors have identified only a few  
64 studies on CHS aluminium beam-columns, namely a relatively old, experimental study by Zhu  
65 and Young (2006), who tested 6082-T6 CHS members subjected to various eccentric  
66 compressive loads, concluding that the European (EC9, 2000), American (Aluminium design  
67 manual, 2000) and Australian/New Zealand (AS/NZS, 1997) design models used for predicting  
68 the capacity of beam-column members produce conservative results. Zhao and Zhai (2019a,  
69 2019b) investigated the performance of SHS and CHS aluminium alloy beam-columns and  
70 proposed a modification for the Chinese design code. More recently, Rong *et al.* (2022b)

71 examined the performance of 7A04-T6 CHS members under eccentric compression and  
72 assessed the European design guidelines, concluding conservative design predictions.  
73 Considering the above, it is deemed necessary to assess the accuracy of some of the  
74 internationally used design models for CHS aluminium alloy beam-columns in order to prevent  
75 inefficient design. Herein focus is placed on CHS beam-columns with stocky cross-sections in  
76 Grades 6082-T6 and 6061-T6 material grades, given that these grades are among the most  
77 widely used in construction. The results of a series of experimental studies on aluminium alloy  
78 CHS members are selected (Section 2) and utilised for the validation of an FE model. The  
79 nonlinear FE model is presented in Section 3 and the validation study in Section 4. In Section  
80 5, a comprehensive parametric study on members with stocky cross-sections is presented,  
81 investigating beam-columns with different lengths, cross-sectional geometries, and material  
82 properties. The numerically obtained strengths are compared with predicted design capacities  
83 in Section 6 to evaluate the codified design provisions of EN 1999-1-1 (2007), the Direct  
84 Strength Method (Schafer, 2008) and Aluminium Design Manual (AA, 2015). Finally, the main  
85 conclusions of this study are presented in Section 7.

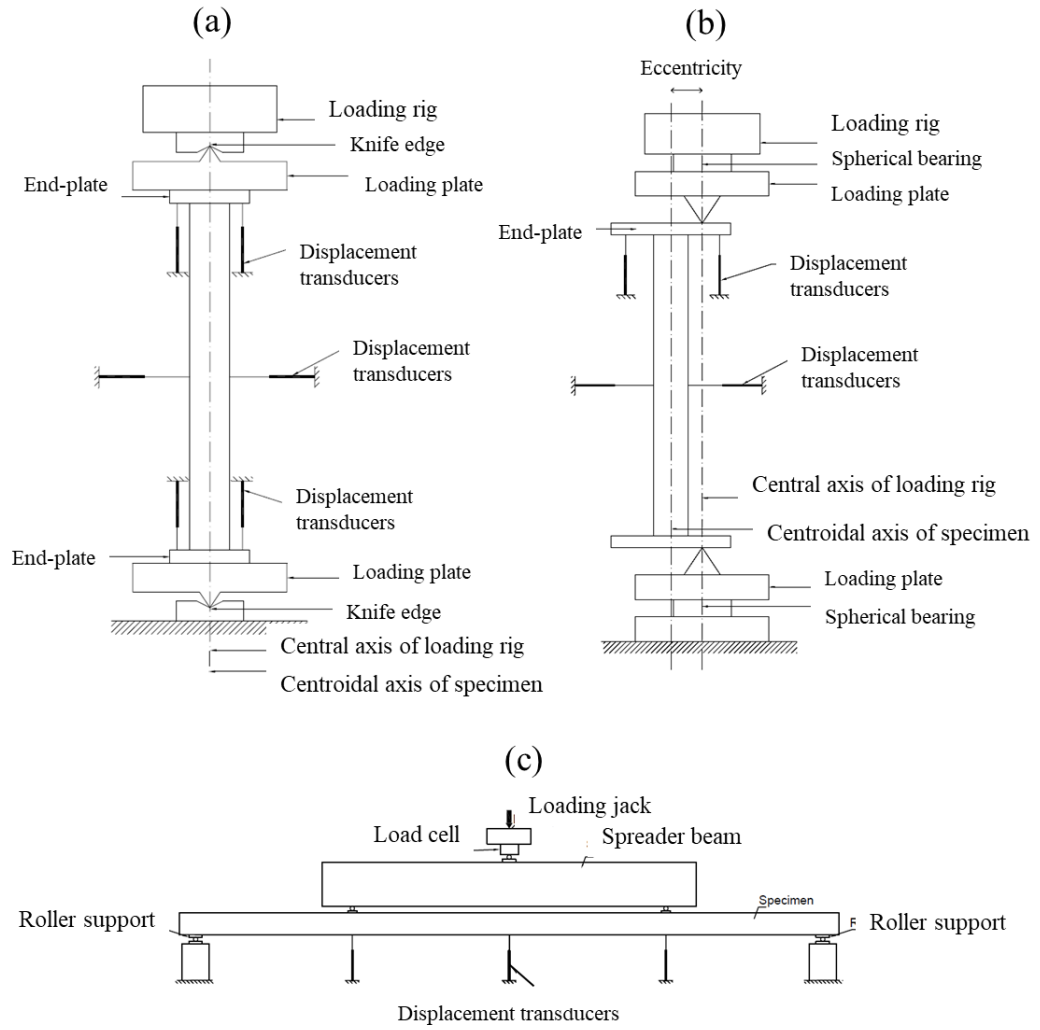
## 86 **2. Experimental database**

87 Experimental and numerical data on aluminium CHS beam-columns are scarce.  
88 Nevertheless, it is critical to ensure that the numerical model is able to accurately capture the  
89 response of two direct loading conditions – axial compression and pure bending, as well as  
90 eccentric loading. For this purpose, the model was validated against four published datasets  
91 published in the literature (Wang et al., 2015; Zhu and Young, 2006; Feng et al., 2020, Zhao  
92 and Zhai, 2020). A total of 34 experiments were employed, comprising 8 columns, 20 beams  
93 and 6 beam-columns, as shown in Table 1, where the specimens' notation is the same as in the  
94 relevant publications. In Figure 1, the experimental set-up which was used in the respective  
95 paper is shown. The main set-up components together with the applied instrumentation during  
96 loading is also shown in this figure. Key test values, such as the main cross-sectional properties  
97 (diameter  $D$ , thickness  $t$ ), the specimen's length ( $L$ ), the eccentricity of the applied load (not  
98 relevant for the beams), the ultimate load ( $N_{u,Exp}$ ) and moment ( $M_{u,Exp}$ ) values, as achieved  
99 during the tests, are listed for reference in Table 2. The selected experimental studies comprised  
100 circular hollow sections made from 6082-T6 and 6061-T6, whose Young's Modulus ( $E$ ), proof  
101 strength ( $\sigma_{0.2}$ ), strain hardening component ( $n$ ) and ultimate strength ( $\sigma_u$ ) are also listed in the

102 same table. These two grades, which are commonly used in structural engineering, will also be  
 103 the focus in the numerical study.

104 Table 1: Tests used for validation purposes.

Reference	Type	Specimen name (based on references)	Material
Wang et al. (2015)	Columns (8 tests)	O89-L500-A	6082-T6
		O89-L800-A	
		O89-L800-C	
		O89-L1200-B	
		O89-L1650-A	
		O89-L1650-B	
		O89-L1650-C	
		O76-L1700-A	
Zhu and Young (2006)		C1 BEAM	6061-T6
	C2 BEAM		
Feng et al. (2020)		HC150x7-B4	6061-T6
Zhao and Zhi (2020)	Beams (20 tests)	CHS-89-4-1800A	6082-T6
		CHS-89-4-1800B	
		CHS-89-4-1800C	
		CHS-89-4-2200A	
		CHS-89-5-1800A	
		CHS-89-5-1800B	
		CHS-89-5-2200A	
		CHS-89-5-2200B	
		CHS-114-4-1800A	
		CHS-114-4-2200A	
		CHS-114-4-2200B	
		HS-114-5.5-1800A	
		CHS-114-5.5-2200A	
		CHS-114-7-1800A	
		CHS-114-7-1800B	
		CHS-114-7-2200A	
CHS-114-7-2200B			
Zhu and Young (2006)	Beam-columns (6 tests)	C1L500E15	6061-T6
		C1L500E40	
		C1L1000E15	
		C1L1000E45	
		C2L1000E15	
		C2L1000E50	



106

107 Figure 1: Experimental set-ups. a) column experiment, adapted from Wang et al., 2015, b)

108 beam-column experiment, adapted from Zhu and Young, 2006, c) beam experiment, adapted

109 from Feng et al., 2020, Zhao and Zhai, 2020.

110

Table 2: Summary of experimental results.

Reference	Specimen name	D (mm)	t (mm)	L (mm)	e (mm)	E (MPa)	$\sigma_{0.2}$ (MPa)	$\sigma_u$ (MPa)	$N_{u,Exp}$ (kN)	$M_{u,Exp}$ (kNm)
Wang et al. (2015)	O89-L500-A	88.80	6.29	500.5	0.693	70233	272.22	(n=25.24)	508.71	-
	O89-L800-A	88.80	6.29	800.0	1.160	70233	272.22	(n=25.24)	444.26	-
	O89-L800-C	88.81	6.29	799.0	0.349	70233	272.22	(n=25.24)	473.67	-
	O89-L1200-B	88.81	6.29	1200.0	1.049	70233	272.22	(n=25.24)	378.68	-
	O89-L1650-A	88.62	6.31	1649.0	6.361	70233	272.22	(n=25.24)	239.67	-
	O89-L1650-B	88.63	6.30	1650.0	1.184	70233	272.22	(n=25.24)	269.84	-
	O89-L1650-C	88.70	6.29	1650.0	1.997	70233	272.22	(n=25.24)	265.55	-
	O76-L1700-A	75.51	2.81	1700.0	2.239	70233	272.22	(n=25.24)	74.17	-
Zhu and Young (2006)	C1 BEAM	49.80	1.59	1000.0	-	67100	286.70	310.10	-	1.15
	C2 BEAM	49.80	3.08	1000.0	-	73400	242.80	262.80	-	2.14
Feng et al. (2020)	HC150x7-B4	202.53	7.57	2699.9	-	69700	198.72	218.64	-	65.60
Zhao and Zhai (2020)	CHS-89-4-1800A	88.62	3.80	1800	-	68190	285.00	(n=28.5)	-	8.24
	CHS-89-4-1800B	88.63	3.88	1800	-	68190	285.00	(n=28.5)	-	8.06
	CHS-89-4-1800C	88.83	3.90	1800	-	68190	285.00	(n=28.5)	-	8.47
	CHS-89-4-2200A	88.41	3.92	2200	-	68190	285.00	(n=28.5)	-	7.66
	CHS-89-5-1800A	88.89	4.97	1800	-	68190	285.00	(n=28.5)	-	11.21
	CHS-89-5-1800B	89.12	5.08	1800	-	68190	285.00	(n=28.5)	-	10.67
	CHS-89-5-2200A	88.82	4.99	2200	-	68190	285.00	(n=28.5)	-	10.63
	CHS-89-5-2200B	88.94	4.99	2200	-	68190	285.00	(n=28.5)	-	11.09
	CHS-114-4-1800A	114.22	3.93	1800	-	68190	285.00	(n=28.5)	-	16.00
	CHS-114-4-2200A	114.42	4.03	2200	-	68190	285.00	(n=28.5)	-	14.17
	CHS-114-4-2200B	113.96	4.06	2200	-	68190	285.00	(n=28.5)	-	14.50
	CHS-114-5.5-1800A	113.84	5.55	1800	-	68190	285.00	(n=28.5)	-	18.93
	CHS-114-5.5-2200A	114.16	5.45	2200	-	68190	285.00	(n=28.5)	-	19.24
	CHS-114-7-1800A	113.99	6.90	1800	-	68190	285.00	(n=28.5)	-	25.61
	CHS-114-7-1800B	113.46	6.76	1800	-	68190	285.00	(n=28.5)	-	24.81
	CHS-114-7-2200A	113.61	6.90	2200	-	68190	285.00	(n=28.5)	-	26.03
	CHS-114-7-2200B	113.84	6.90	2200	-	68190	285.00	(n=28.5)	-	24.83
	CHS-89-4-1800A	88.62	3.80	1800	-	68190	285.00	(n=28.5)	-	8.240
Zhu and Young (2006)	C1L500E15	50.00	1.57	500.1	14.6	67100	286.70	310.10	29.1	0.569
	C1L500E40	50.10	1.56	500.0	42.7	67100	286.70	310.10	16.8	0.894
	C1L1000E15	50.00	1.55	1000.4	17.8	67100	286.70	310.10	19.9	0.912
	C1L1000E45	50.10	1.55	1000.0	46.9	67100	286.70	310.10	12.0	0.931
	C2L1000E15	49.70	3.05	1000.0	16.0	67100	286.70	310.10	36.0	1.469
	C2L1000E50	49.80	3.04	1000.0	50.5	67100	286.70	310.10	20.0	1.727

112

### 113 3. Numerical Modelling Assumptions

114 The finite element models of the aluminium alloy CHS specimens were developed using the  
115 commercial software package Abaqus (2021). Initially, the midline dimensions of the cross-  
116 section were modelled, in line with previously published studies on thin-walled metallic  
117 sections (Wang et al, 2016; Gkantou et al., 2017a). Four node double-curved shell elements  
118 with reduced integration were employed to avoid shear and membrane locking of models.  
119 Subsequently, the specimens were extruded to their desired lengths, as shown in Figure 2. The

120 material response which indicated rounded response (was approximated based on Ramberg and  
121 Osgood (1943) model and was modelled with the Von Mises yield criterion with isotropic  
122 hardening, using the elastic and plastic material properties reported in the respective  
123 experimental studies.

124 The adopted boundary conditions aimed to simulate those presented in Figure 1, for each  
125 test type. For the column and the beam-column members, the top of the specimens can rotate  
126 and longitudinally shorten, whereas the bottom is modelled as pinned. By assigning the  
127 boundary conditions and tying the cross-section end nodes to a reference point (RP-TOP, RP-  
128 BOTTOM), the rotational point of both knife edges in the experimental testing can be  
129 modelled, ensuring the numerical analysis can capture an accurate response of the  
130 deformations. The load is then applied to the top reference point, as shown in Figure 2. Contrary  
131 to the columns, where the axial compressive load is applied concentrically, in the beam-column  
132 models, the compressive load is applied with an eccentricity from the cross-sectional centroid,  
133 as reported in Table 2.

134 The development of the FE models for the four-point bending tests was similar to that of  
135 the column and beam-column members in terms of chosen element type and material model.  
136 Simply supported beam boundary conditions restricted displacement in the x and y-axis (U1  
137 and U2) at the support location (see Figure 2(c)). Following the reported testing procedures,  
138 steel stiffeners were used to reinforce the areas susceptible to local bearing failure located at  
139 the support and loading points and were included in the numerical analysis by utilising  
140 kinematic coupling constraints. This ensured the cross-section was rigid at the support and  
141 loading locations, whilst applying the assigned boundary conditions across the whole cross-  
142 section. According to the reported experimental studies, the beam was subjected to pure  
143 bending in the central moment region.

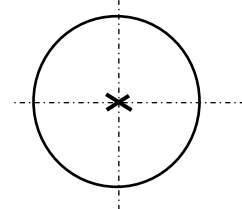
144 Initial geometric imperfections can affect the buckling response of thin-walled metallic  
145 structures. A linear perturbation buckling analysis is initially used to extract the eigenvalues.  
146 The lowest local and global buckling mode shapes are imported as initial geometric  
147 imperfections in the subsequent nonlinear analysis, which is executed with the general static  
148 Riks method. The amplitude of the required initial geometric imperfection size to accurately  
149 simulate the members considered is determined based on a sensitivity analysis reported in the  
150 next section.

151



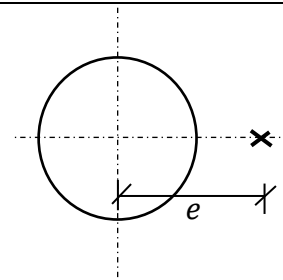
- Rigid body at RP-TOP
- BC at RP-TOP:  
 $u_x = u_y = u_{rz} = 0, u_z \neq 0, u_{rx} \neq 0, u_{ry} \neq 0$
- Downwards vertical load at RP-TOP

- Rigid body at RP-BOTTOM
- BC at RP-BOTTOM:  
 $u_x = u_y = u_z = u_{rz} = 0, u_{rx} \neq 0, u_{ry} \neq 0$



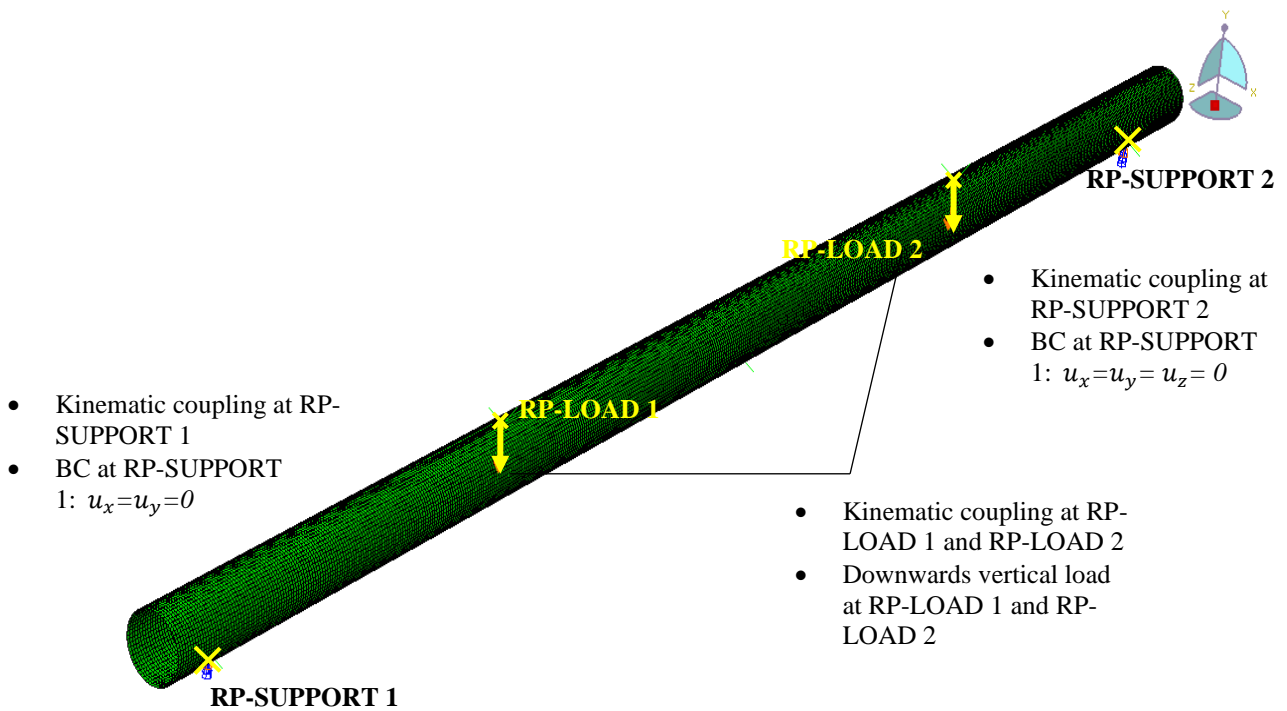
RP-TOP  
 RP-BOTTOM } At centroid

(a) columns



RP-TOP  
 RP-BOTTOM } At distance  $e$  (eccentricity) from centroid

(b) beam-columns



- Kinematic coupling at RP-SUPPORT 1
- BC at RP-SUPPORT 1:  $u_x = u_y = 0$

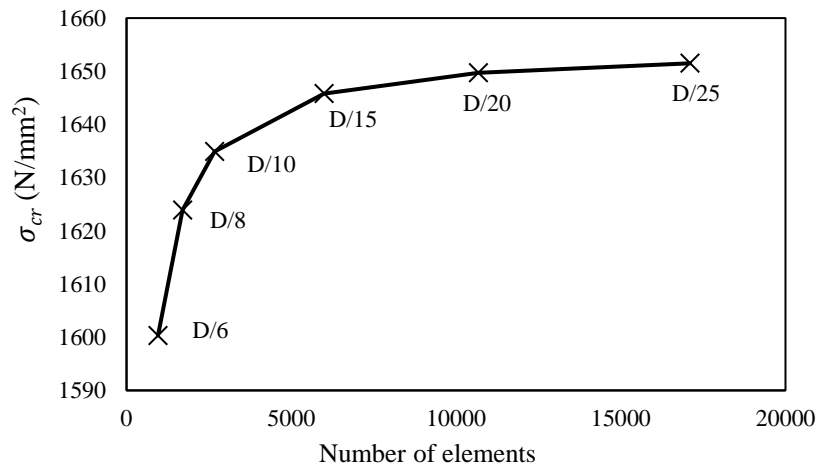
- Kinematic coupling at RP-SUPPORT 2
- BC at RP-SUPPORT 2:  $u_x = u_y = u_z = 0$

- Kinematic coupling at RP-LOAD 1 and RP-LOAD 2
- Downwards vertical load at RP-LOAD 1 and RP-LOAD 2

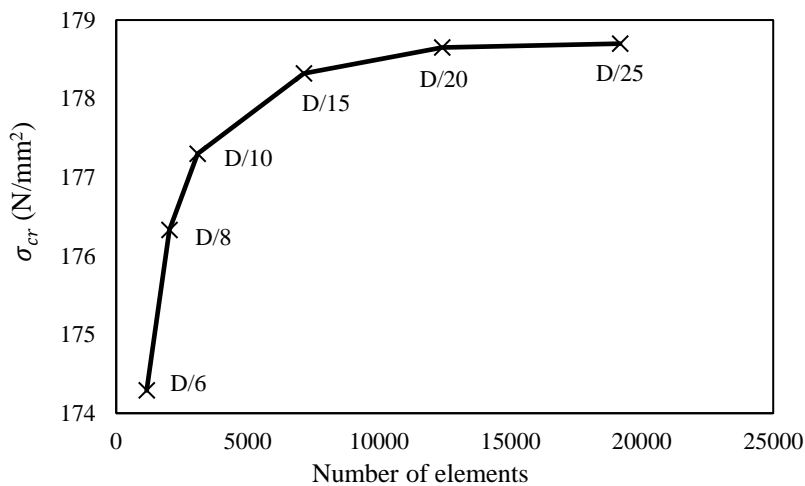
(c) beams

Figure 2: Numerical Models.

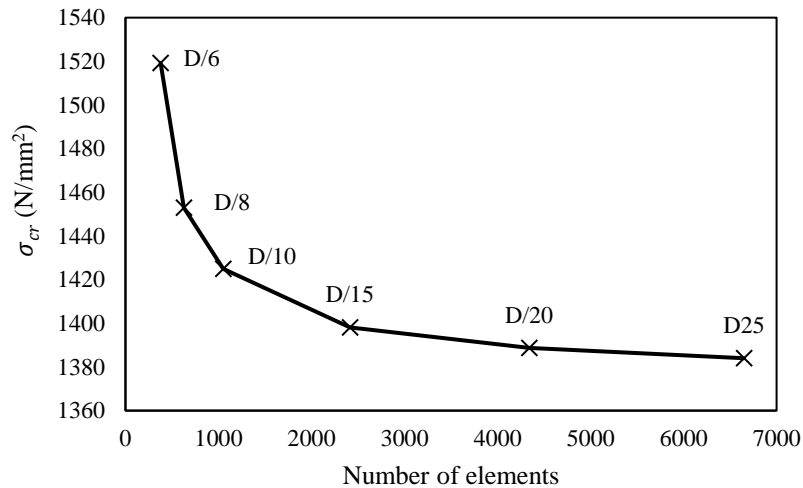
154 To optimise the computational time required for the simulations whilst ensuring the results  
 155 of the analysis are not affected by mesh size, a mesh convergence study was also performed on  
 156 the models. Six different mesh configurations, with average element size given as a function  
 157 of the cross-section's diameter ( $D/6$ ,  $D/8$ ,  $D/10$ ,  $D/15$ ,  $D/20$ ,  $D/25$ ), were examined and the  
 158 elastic critical buckling stress,  $\sigma_{cr}$ , was calculated. Once the difference between two  
 159 consecutive mesh sizes was below 1%, it was concluded that a further decrease of the mesh  
 160 size would not contribute to increased accuracy. Typical examples from the mesh convergence  
 161 study for a numerically modelled column, beam-column and beam are shown in Figure 3.  
 162 Overall, it was found that  $D/20$  was the optimum mesh size and therefore it was used for further  
 163 analysis in Sections 4 and 5.



(a) columns (O89-L800-A)



(b) beam-columns (C1L500E15)



(c) beams (HC150x7-B4)

164 Figure 3: Mesh study - typical cases for each studied member.

165 Finally, the residual stresses have been found to have little influence on the load-carrying  
 166 capacities of extruded aluminium alloy profiles (Mazzolani, 1995). In line with past studies  
 167 (Feng et al., 2019; Georgantzia et al., 2023), they were not explicitly included in the FE models  
 168 developed herein. Nevertheless, very good agreement between experimental and numerical  
 169 performance has been achieved, as will be shown in the following section.

#### 170 4. Validation of the FE models

171 The numerical models were validated by comparing the experimentally obtained ultimate  
 172 loads and moments against the corresponding numerical values (Table 3), the overall load-  
 173 deflection curves (Figure 4) and failure modes (Figure 5). A geometric imperfection sensitivity  
 174 study was initially performed on the FE models to help identify which local and global  
 175 geometric imperfection amplitudes obtained the best agreement between the experimental data  
 176 and numerical results. Four different combinations of imperfection amplitudes were used in the  
 177 imperfection study, for the columns and beam-columns, considering imperfection amplitudes  
 178 as fractions of the section thickness and member length for buckling mode shapes pertinent to  
 179 local and global buckling respectively:  $t/10 + L/1000$ ,  $t/10 + L/1500$ ,  $t/100 + L/1000$ ,  $t/100 +$   
 180  $L/1500$ . For the beam models, two local geometric imperfection amplitudes were examined:  
 181  $t/10$ , and  $t/100$ . The comparison of the experimental-to-numerical capacity values are shown  
 182 in Table 3. When comparing the ultimate axial load ratios ( $N_{u,Exp}/N_{u,FE}$ ) for the columns and  
 183 beam-columns specimens, it was found that the imperfection amplitude  $t/10 + L/1000$  led to  
 184 the best agreement between the experimental and numerical ultimate capacities, achieving a

185 mean ratio of 0.98 and 0.96, respectively. Whilst the local geometric amplitude of  $t/10$  did not  
 186 lead to the best agreement between the experimental beam results and the corresponding FE  
 187 predictions, overall it provided a successful validation and hence it was deemed accurate  
 188 enough and will be used in the subsequent parametric study. In Figure 4, typical load-deflection  
 189 curves are presented, showing a successful replication of the experimental response by the  
 190 numerical models. Finally, successfully validated failure modes are depicted in Figure 5.

191 Table 3: Comparison of experimental and numerical ultimate response including an  
 192 imperfection sensitivity study.

193 (a) columns

Specimen	Imperfection Magnitude			
	$t/10 + L/1000$	$t/10 + L/1500$	$t/100 + L/1000$	$t/100 + L/1500$
	$N_{u,Exp}/N_{u,FE}$	$N_{u,Exp}/N_{u,FE}$	$N_{u,Exp}/N_{u,FE}$	$N_{u,Exp}/N_{u,FE}$
C1L500E15	0.99	0.99	0.97	0.97
C1L500E40	0.98	0.98	0.95	0.95
C1L1000E15	1.00	1.00	1.00	1.00
C1L1000E45	0.96	0.96	0.96	0.96
C2L1000E15	1.00	1.00	0.99	0.99
C2L1000E50	0.96	0.96	0.96	0.96
Mean	0.98	0.98	0.97	0.97
COV	0.02	0.02	0.02	0.02

194

195 (b) beam-columns

Specimen	Imperfection Magnitude			
	$t/10 + L/1000$	$t/10 + L/1500$	$t/100 + L/1000$	$t/100 + L/1500$
	$N_{u,Exp}/N_{u,FE}$	$N_{u,Exp}/N_{u,FE}$	$N_{u,Exp}/N_{u,FE}$	$N_{u,Exp}/N_{u,FE}$
O89-L500-A	1.01	1.01	0.99	0.99
O89-L800-A	0.97	0.96	0.96	0.96
O89-L800-C	1.01	1.00	1.01	1.00
O89-L1200-B	1.11	1.09	1.11	1.09
O89-L1650-A	0.82	0.81	0.81	0.80
O89-L1650-B	0.94	0.91	0.94	0.91
O89-L1650-C	0.95	0.93	0.95	0.93
O77-L1700-A	0.85	0.82	0.85	0.81
Mean	0.96	0.94	0.95	0.94
COV	0.10	0.10	0.10	0.10

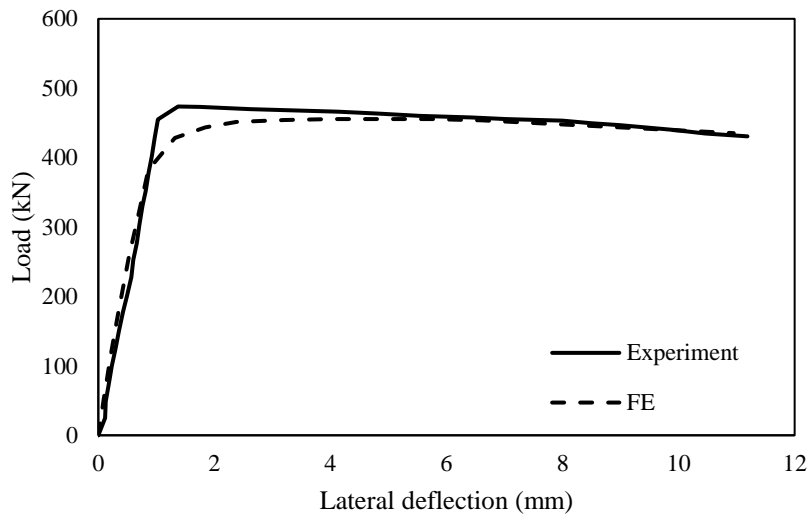
196

197 (c) beams

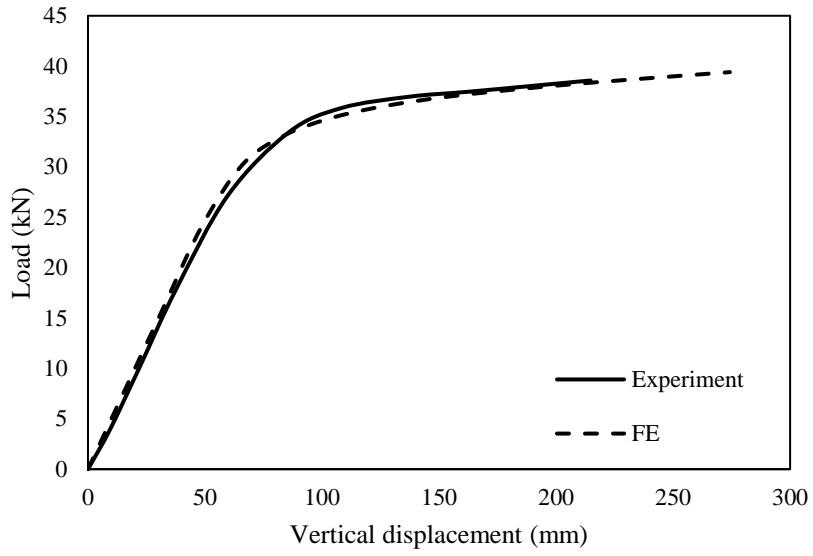
Specimen	Imperfection Magnitude	
	$t/10$	$t/100$
	$M_{u,Exp}/M_{u,FE}$	$M_{u,Exp}/M_{u,FE}$

C1 BEAM	1.14	1.11
C2 BEAM	1.07	1.04
HC150x7-B4	1.01	0.99
CHS-89-4-1800A	0.99	0.99
CHS-89-4-1800B	1.00	0.96
CHS-89-4-1800C	1.00	1.00
CHS-89-4-2200A	0.91	0.91
CHS-89-5-1800A	1.03	1.02
CHS-89-5-1800B	1.00	0.96
CHS-89-5-2200A	1.00	0.99
CHS-89-5-2200B	1.02	1.02
CHS-114-4-1800A	1.15	1.15
CHS-114-4-2200A	1.03	0.99
CHS-114-4-2200B	1.06	1.01
CHS-114-5.5-1800A	1.01	0.97
CHS-114-5.5-2200A	1.01	1.00
Mean	1.03	1.02
COV	0.05	0.05

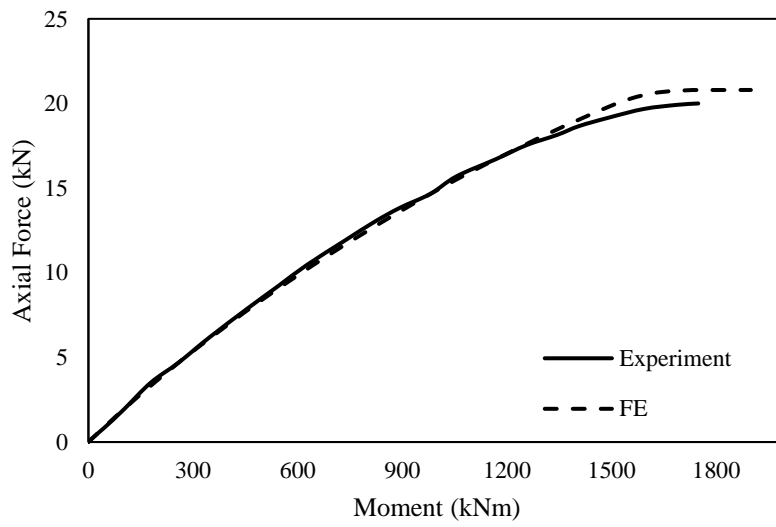
198



(a) columns (O89-L800-A)



(b) beams (CHS 89-5-2200-A)

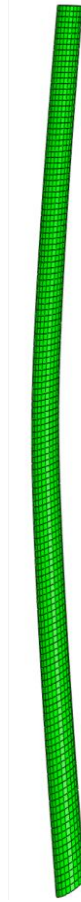


(c) beam-columns (C2L1000E50)

199

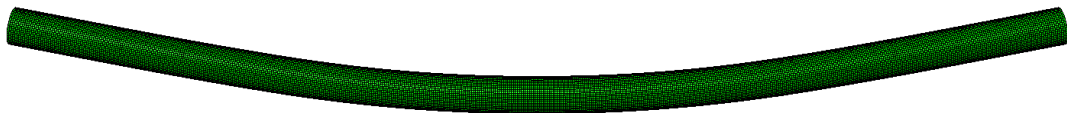
Figure 4: Comparison of experimental and numerical curves for typical cases.

200



(a) columns (089-L1650-A)

(b) beam-columns (C1L1000E45)



(c) beams (CHA-89-5-2200B)

201

Figure 5: Comparison of experimental and numerical failure modes.

202

203 **5. Parametric study**

204 Following the successful validation of the non-linear FE model, a parametric study was  
 205 conducted on 168 aluminium CHS models with stocky cross-sections. Three cross-sections,  
 206 four lengths, six eccentricities and a pure bending case, as presented in Table 4, were modelled  
 207 for each of the two different aluminium alloys (6082-T6 and 6061-T6). Average material  
 208 properties from those reported in the chosen experimental studies were used for the parametric  
 209 study, as shown in Figure 6. The member lengths were chosen to obtain a range of member  
 210 slendernesses, whilst the external diameter was kept constant and equal to 50 mm and the  
 211 thickness was varied from three to five mm with a step of 1 mm. The eccentricities were varied  
 212 from zero to 100 mm to provide a range of axial load to bending moment ratios to evaluate the  
 213 existing design interaction curves for members subjected to a combined axial load and moment  
 214 (Section 6). The validation results indicated that local and global geometrical imperfection  
 215 amplitudes of  $t/10$  and  $L/1000$ , respectively, led to the most accurate results and were therefore  
 216 used for the parametric study of the column and beam-column members. In line with the  
 217 considerations described in Section 4, a local imperfection amplitude of  $t/10$  that ensured an  
 218 adequate capture of the ultimate load and moment-curvature response was chosen for the beam  
 219 specimens. These amplitudes have been identified to produce accurate numerical results and  
 220 have been adopted in previously published numerical studies on thin-walled members made of  
 221 nonlinear metallic materials, including Wang et al. (2016) and Zhu and Young (2008).

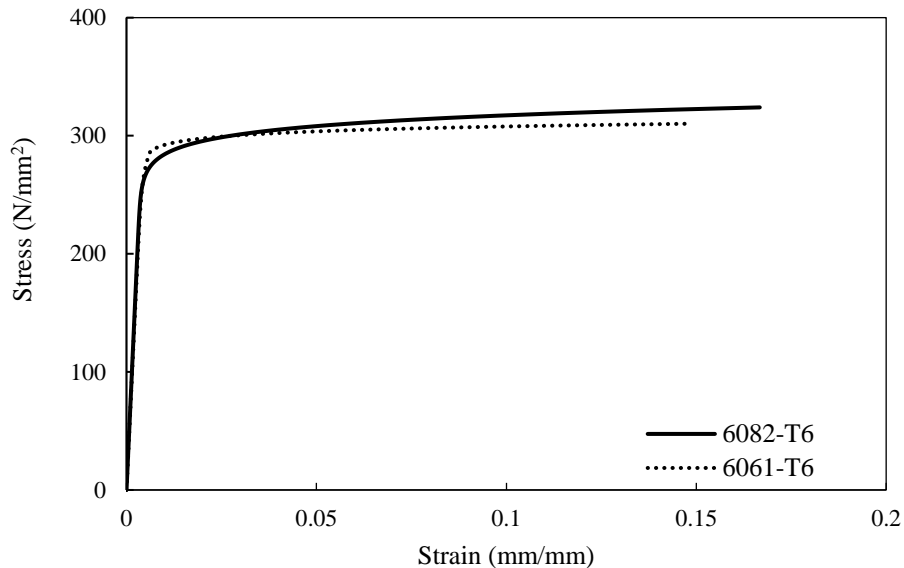
222

223 Table 4: List of key parameters included in the parametric study.

Total analyses: 168	
3 cross-sections	<ul style="list-style-type: none"> <li>• 50×3</li> <li>• 50×4</li> <li>• 50×5</li> </ul>
2 aluminium alloy materials	<ul style="list-style-type: none"> <li>• 6082-T6</li> <li>• 6061-T6</li> </ul>
4 lengths	<ul style="list-style-type: none"> <li>• 300 mm</li> <li>• 600 mm</li> <li>• 900 mm</li> <li>• 1200 mm</li> </ul>
7 loading cases	<ul style="list-style-type: none"> <li>• e: 0 mm (column)</li> <li>• e: 2 mm (beam-column)</li> <li>• e: 5 mm (beam-column)</li> </ul>

- 
- e: 10 mm (beam-column)
  - e: 25 mm (beam-column)
  - e: 100 mm (beam-column)
  - beam
- 

224



225

226

Figure 6: Stress-strain curves used for the parametric study.

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A typical example of the obtained load-deflection curves is shown in Figure 7, where the response of a specimen made from 6061-T6 for all four studied lengths is depicted. Note that the labels used in the legend correspond to the type of applied loading: “C” for column and “BC” for beam-column, followed by thickness, length and applied eccentricity. For example, BC-4-300-25 denotes a beam-column member of cross-section 50×4 mm, length of 300 mm and applied eccentricity of 25 mm. Overall the following observations can be made for the parameters studied:

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- Material grade: the study focussed on two 6xxx series aluminium alloys with T6 heat treatment that presented similar material response, as shown in Figure 6. This has led to similar performance of the two studied alloys, as will also be quantitatively demonstrated in Section 6.
- Member length: As anticipated, the ultimate load decreased with increasing member length, for example the ultimate capacity of 300 mm member with  $e=10$  mm is equal to 100.54 kN (see Figure 7(a)), whilst it falls to 59.41 kN and (see Figure 7(c)) for a 900 mm member with the same eccentricity.

- 242 Applied eccentricities: Based on Figures 7(a)-7(d), it can generally be observed that

243 the specimens subjected to small loading eccentricities (0, 2, 5, 10 mm) reached their

244 peak load with minimal deflections before a rapid loss of stiffness occurred, causing

245 extensive deformations as the load reduced due to the member failing by flexural

246 buckling. Specimens subjected to a larger eccentric load (e.g., 25, 100 mm) generally

247 reached their peak loads much later than their counterparts and presented a smoother

248 load descending curve for increasing deformations.
- 249 Cross-sections: The study included only stocky cross-sections that are not prone to

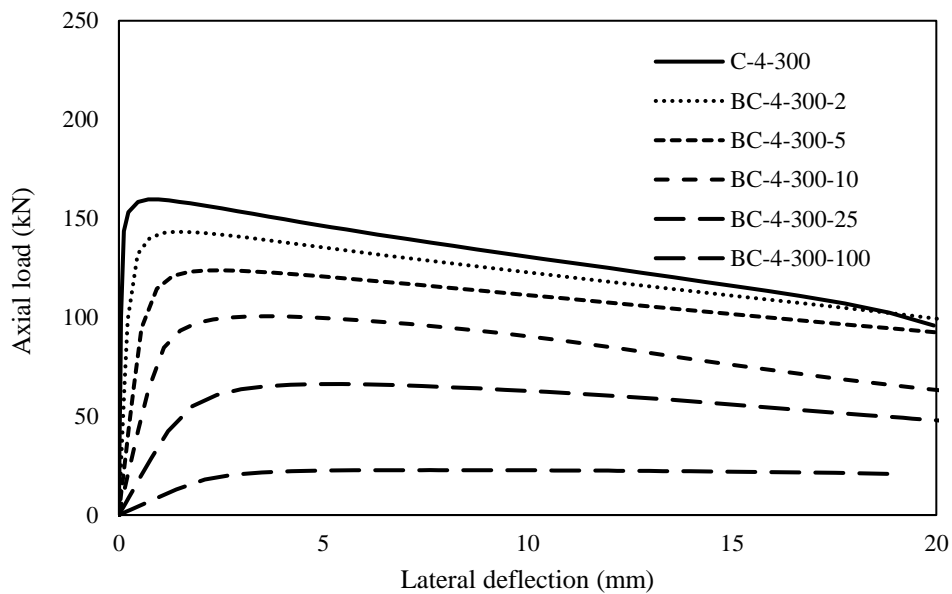
250 local buckling effects. As anticipated, higher load and moment capacities have been

251 observed in cross-sections with larger thickness. For example, the member BC-5-

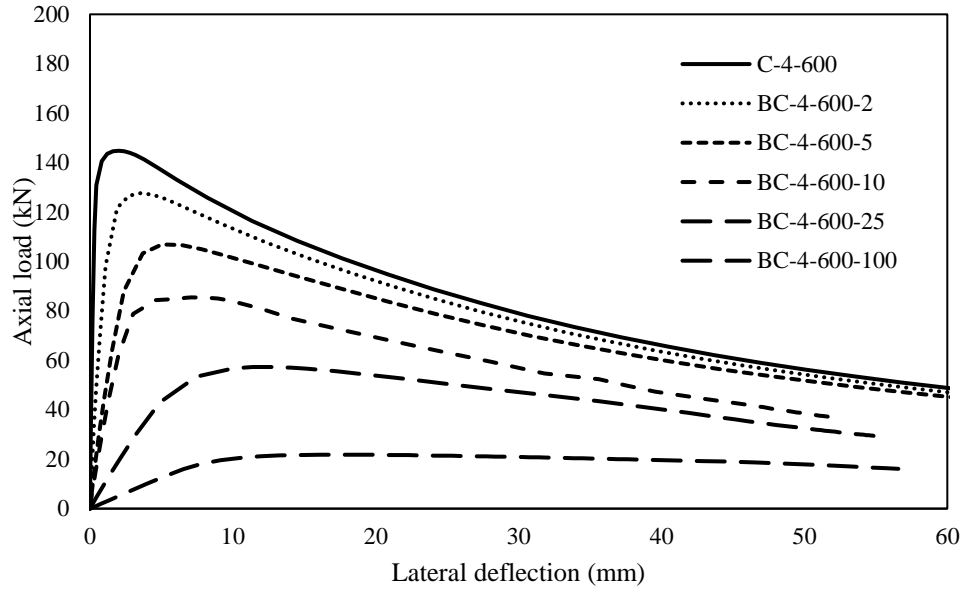
252 300-25 of 6082-T6 presented ultimate load equal to 76.9 kN, whilst for the same

253 member of 3 mm thickness (BC-3-300-25), the ultimate numerically achieved load

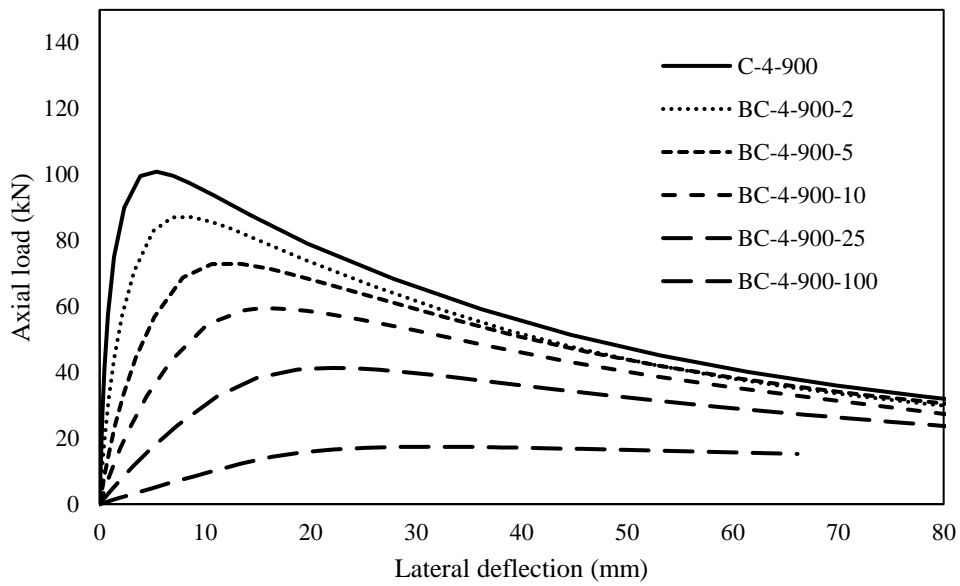
254 was equal to 49.2 kN.



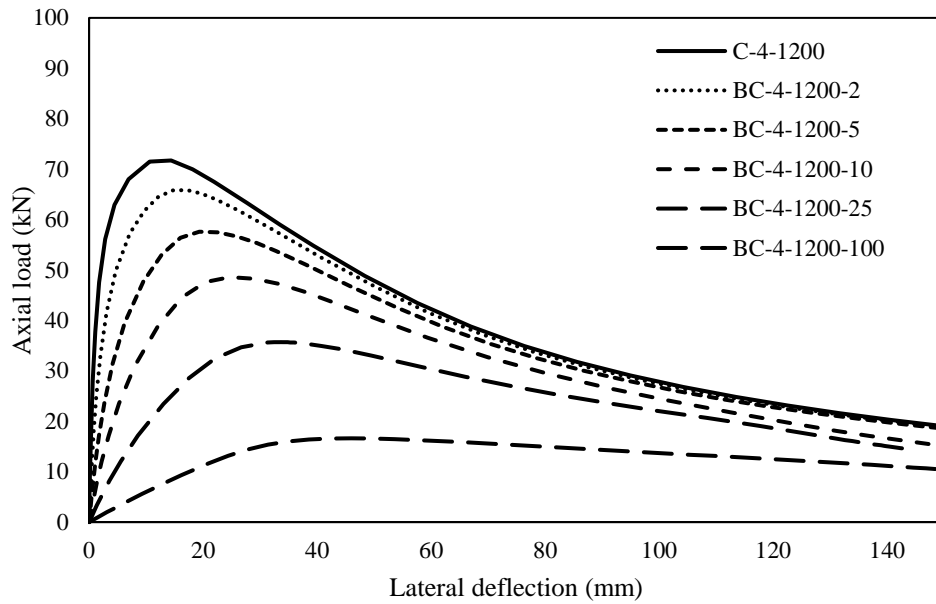
(a) Length: 300 mm



(b) Length: 600 mm



(c) Length: 900 mm



(d) Length: 1200 mm

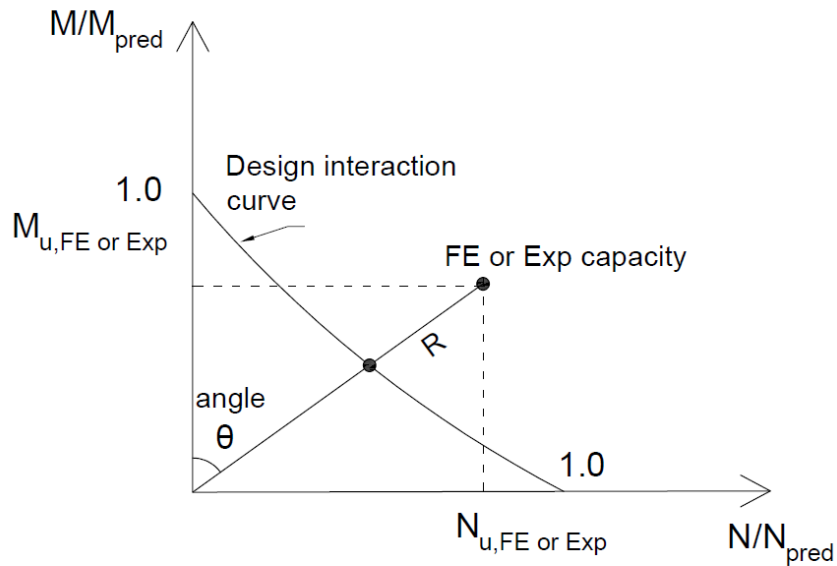
255 Figure 7: Typical parametric study results (material: 6061-T6, cross-section: 50×4).

256

## 257 6. Assessment of design guidelines

258 The numerically obtained capacities are utilised in this section to assess and compare the  
 259 accuracy of the following design predictions: European design provisions, Direct Strength  
 260 Method, Aluminium Design Manual. All safety factors are set to unity. The design strengths  
 261 for both 6061-T6 and 6082-T6 aluminium alloy beam-columns were calculated using the  
 262 predicted strengths ( $M_{pred}$ ,  $N_{pred}$ ) and then compared with the numerically obtained ones. In  
 263 order to allow for a more direct assessment, the ratio  $R$  of the beam-column specimens can  
 264 be established. In Figure 8, the bending moment capacity normalised with the predicted one  
 265 ( $M/M_{pred}$ ) is plotted against the axial compressive capacity normalised with the predicted one  
 266 ( $N/N_{pred}$ ), with both ratios being equal to unity at their intersection with the respective axis.  
 267 In the same figure, it is also included the design interaction curve that links the two extremes  
 268 (i.e., from pure bending at vertical axis to pure compression at horizontal axis), as well as a  
 269 FE result ( $N_u$ ,  $M_u$ ). Note that for test/FE points above the design interaction curve indicate an  
 270 overconservative design estimation; points below the design curve indicates unsafe  
 271 predictions, whilst highest accuracy in predicted capacity is achieved when the test/FE points  
 272 are closer to the curve. The capacity ratio  $R$  which is used in the assessments hereafter,  
 273 represents the distance from the upper end of the straight line extending from the origin (0,0)  
 274 to ( $N_u$ ,  $M_u$ ) and the intersection of this line with the design interaction curve. The angle ( $\theta$ )

275 can also be defined to indicate the ratio of the combined axial compression and bending  
 276 moment applied to the specimen (the two extremes being:  $\theta = 0^\circ$ , which shows that the  
 277 specimen is subjected to pure bending, and  $\theta = 90^\circ$ , which shows that the specimen is  
 278 subjected to axial compression). In addition to the 168 FE data of the parametric study shown  
 279 in Section 5, the assessment hereafter utilises the 34 experimental data, reported at Wang et  
 280 al. (2015), Zhu and Young (2006), Feng et al. (2020) and Zhao and Zhai (2020).



281

282 Figure 8: Design interaction curve – definition of R and  $\theta$ .

### 283 6.1 European design provisions

284 In order to estimate the resistance of aluminium alloy beam-column members, EN 1999-1-  
 285 1 (2007) suggests an interaction formula between the buckling resistance and bending  
 286 resistance of a member.

287 The calculation of the buckling resistance ( $N_{EC9}$ ) of an aluminium alloy member is defined  
 288 by Equation (1)

$$N_{EC9} = \chi A \sigma_{0.2} \quad (1)$$

289 Where  $\chi$  is the buckling reduction factor,  $A$  is the gross area of the cross-section, and  $\sigma_{0.2}$  is the  
 290 characteristic yield/proof strength of the material. The reduction coefficient  $\chi$  is given by  
 291 Equation (2).

$$\chi = \frac{1}{\varphi + [\varphi^2 - \bar{\lambda}^2]^{0.5}} \leq 1.0 \quad (2)$$

292 The parameter  $\varphi$  can be estimated by Equation (3)

$$\varphi = 0.5 \left[ 1 + \alpha(\bar{\lambda} - \bar{\lambda}_0) + \bar{\lambda}^2 \right] \quad (3)$$

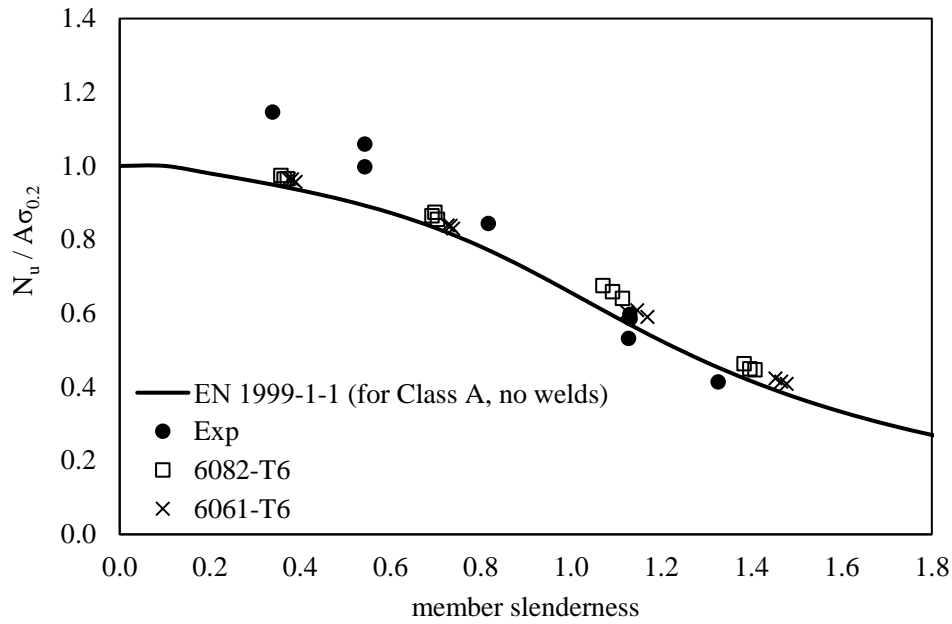
293 where  $\alpha$  and  $\bar{\lambda}_0$  is the imperfection factor and the limit of the horizontal plateau, respectively,  
 294 corresponding to the relevant buckling curve (for 6082-T6, 6061-T6:  $\alpha$  is 0.2 and  $\bar{\lambda}_0$  is 0.1) The

295 member slenderness  $\bar{\lambda}$  is given by Equation (4)

$$\bar{\lambda} = \sqrt{\frac{A\sigma_{0.2}}{N_{cr}}} \quad (4)$$

296 where  $N_{cr}$  is the elastic critical buckling load of the column, based on the gross cross-sectional  
 297 properties. Note that all considered cross-sections were Class 1-2, based on the slenderness  
 298 limits specified in EC9, and thus the remaining classes are out of the scope of the present study.

299 Using the 24 FE data of the members subjected to pure compression and 8 experimental  
 300 data from Wang et al. (2015), the accuracy of the Eurocode buckling curve (i.e., Equations (2)-  
 301 (3)) can be assessed. In Figure 9, the ultimate loads ( $N_u$ ) are normalised by the squash load  
 302 ( $A\sigma_{0.2}$ ) and plotted against the member slenderness  $\bar{\lambda}$ . As can be seen, the EC9 buckling curve  
 303 appears to generally predict well the buckling loads.



304

305

Figure 9: Assessment of EC9 buckling curve.

306

For the calculation of the bending resistance ( $M_{EC9}$ ) of the members, EC9 specifies Equation

307 (5):

$$M_{EC9} = \alpha \sigma_{0.2} W_{el} \quad (5)$$

308

Where  $\alpha$ , is the shape factor and  $W_{el}$  is the elastic modulus of the cross-section.

309

The predicted axial and flexural resistances of the members ( $N_{EC9}$  and  $M_{EC9}$ ) were used to produce the normalised axial load and bending moments that will be compared to the interaction curve (see Equation (6)):

311

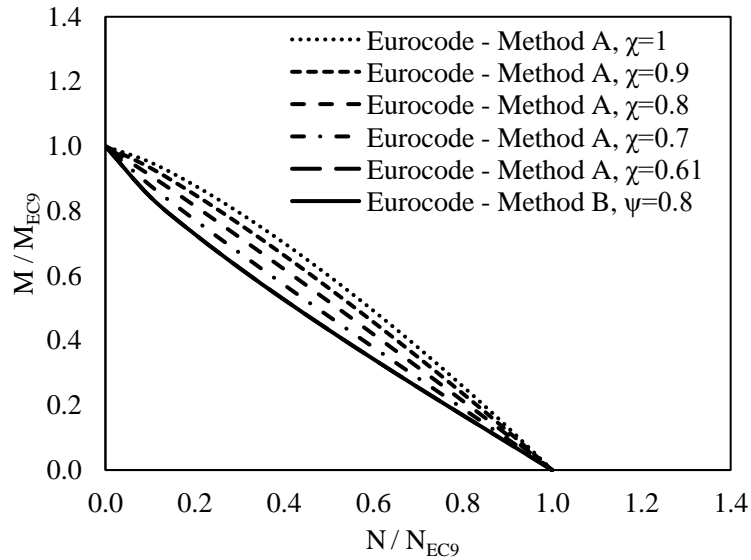
$$\left[ \left( \frac{N_u}{N_{EC9}} \right)^{\psi_c} + \left( \frac{M_u}{M_{EC9}} \right)^{1.7} \right]^{0.6} \leq 1.00 \quad (6)$$

312

Note that EC9 provides two alternative procedures for the evaluation of parameter  $\psi_c$ :  $\psi_c$  (which should be  $\geq 0.8$ ) equals to either  $1.3\chi$  for Class 1 and Class 2 cross-sections and  $\psi_c = \psi (=1.00)$  for Class 3 and Class 4 sections (Method A, hereafter), or alternatively,  $\psi_c$  may be taken as 0.8 for all cross-sections (Method B, hereafter). All relevant interaction curves are presented for comparison purposes in Figure 10, where it can be seen that for Method A, the curve varies based on the reduction parameter  $\chi$  (i.e., based on the member's slenderness),

317

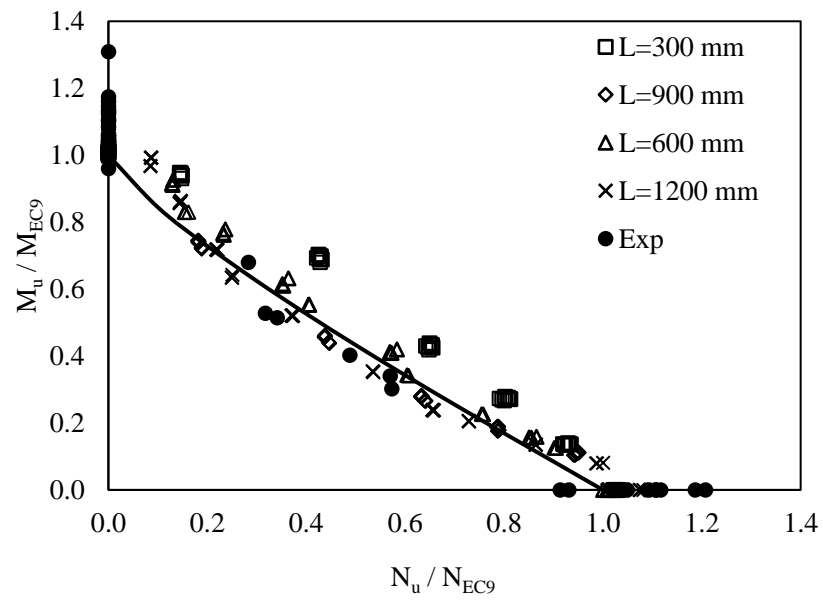
318 whilst for  $\chi \geq 0.61$  the curve coincides with that of Method B, where  $\psi_c$  is always considered  
 319 equal to 0.8.



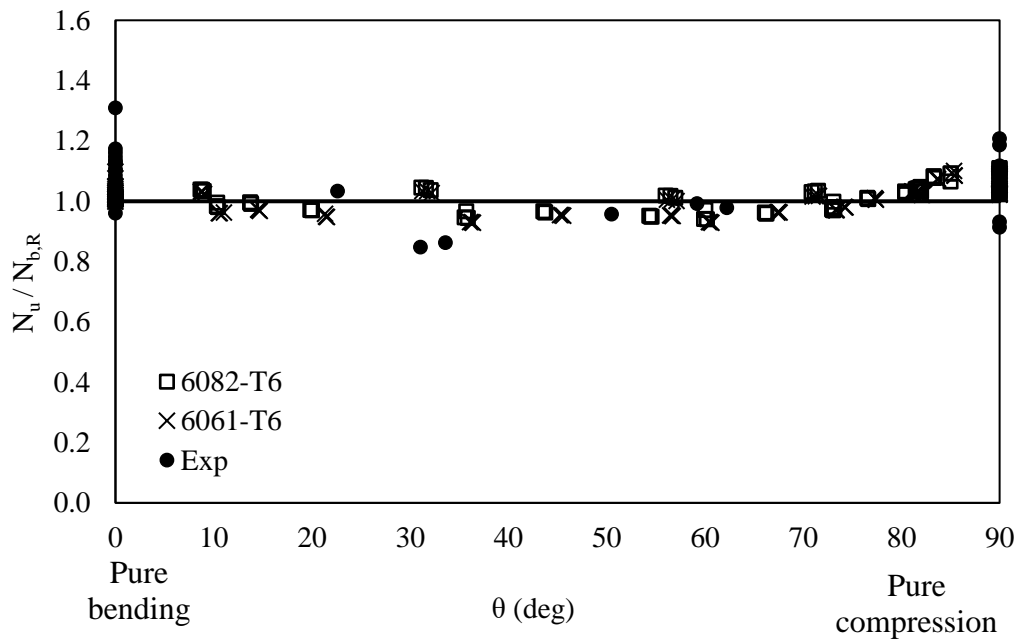
320

321 Figure 10: Eurocode curves (Method A:  $\psi_c$  as a function of  $\chi$  for Classes 1-2, Method B:  
 322  $\psi_c=0.8$  for all Classes)

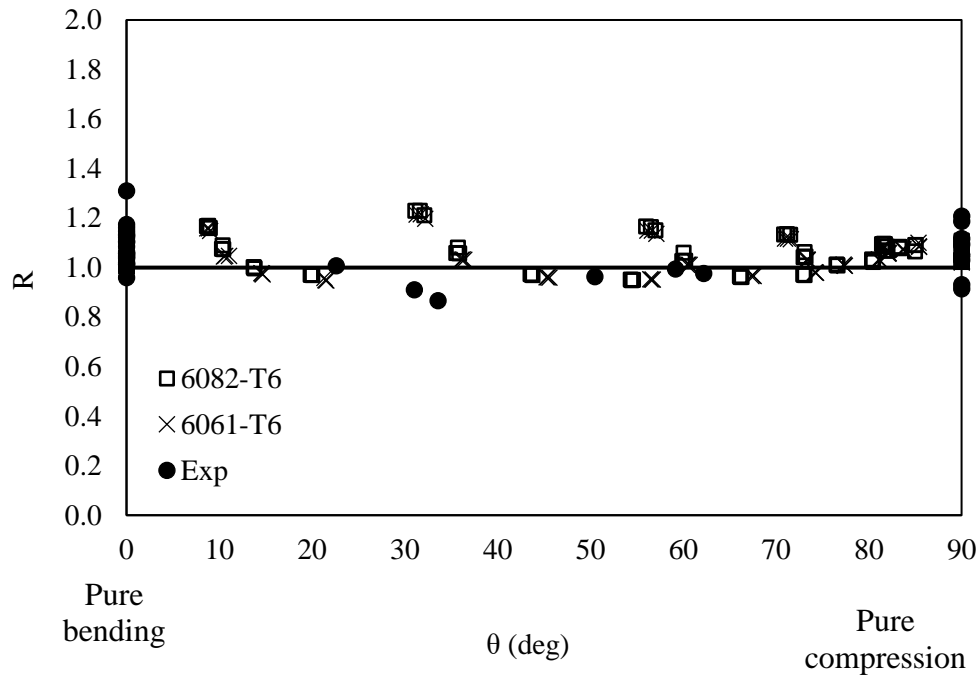
323 In Figure 11, the ultimate performance is normalised by the resistances predicted by EC9  
 324 and plotted in the form of M-N interaction curves. In the same graph, the Eurocode curve with  
 325 Method B (which is constant for all studied specimens) is included. It should be noted that data  
 326 points slightly above the interaction curve show safe predictions. Moving further above the  
 327 line, predictions are conservative, whilst points below the curve correspond to unsafe  
 328 predictions. As shown, the data points lie fairly close to the curve with satisfactory consistency,  
 329 demonstrating accurate predictions. Similar conclusions can be drawn from Figure 12 and  
 330 Table 5, where the results are also presented for the two studied materials, but also separately  
 331 for Methods A and B. It can also be observed that for Method A, for which the parameter  $\psi_c$  is  
 332 a function of the slenderness parameter, the accuracy improves (mean: 1.01 for Method A,  
 333 mean: 1.05 for Method B). In Table 5, the results are also reported based on the considered  
 334 lengths. The predicted capacity ratios appear more accurate for Method A compared to Method  
 335 B for specimens of lengths 300 mm and 600 mm, but are the same for longer specimens. This  
 336 is in line with the limitation of Method A to have  $\psi_c \geq 0.8$  (which corresponds to  $\chi \geq 0.61$ ), that  
 337 leads to same accuracy values for specimens of length 900 mm and 1200 mm. The trend is  
 338 similar for both examined aluminium alloy grades.



339 Figure 11: Ultimate performance normalised with EC9 strength for the two numerically  
 340 studied materials (Design Interaction curved based on Method B).



a) EC9 – Method A



b) EC9 – Method B

341 Figure 12: Assessment of Eurocode; results presented with respect to degree  $\theta$  of  
 342 compression/bending.

343 Table 5: Assessment of Eurocode 9 - Accuracy R; for the two studied materials.

Specimen	No FE/exp	Eurocode - Method A				Eurocode - Method B			
		6082-T6		6061-T6		6082-T6		6061-T6	
		mean	COV	mean	COV	mean	COV	mean	COV
C/BC/B-3-300	14	1.03	0.01	1.02	0.01	1.11	0.07	1.10	0.06
C/BC/B -4-300	14	1.03	0.01	1.02	0.01	1.12	0.07	1.11	0.06
C/BC/B -5-300	14	1.03	0.01	1.02	0.01	1.12	0.07	1.04	0.02
C/BC/B -3-600	14	0.99	0.04	0.98	0.04	1.04	0.02	1.03	0.02
C/BC/B -4-600	14	1.01	0.03	0.98	0.04	1.06	0.03	1.03	0.02
C/BC/B -5-600	14	0.99	0.04	0.98	0.05	1.04	0.02	1.03	0.01
C/BC/B -3-900	14	1.02	0.06	1.00	0.05	1.02	0.05	1.01	0.04
C/BC/B -4-900	14	1.02	0.06	1.01	0.05	1.02	0.05	1.02	0.05
C/BC/B -5-900	14	1.02	0.05	1.02	0.06	1.02	0.05	1.02	0.06
C/BC/B-3-1200	14	1.01	0.05	1.02	0.06	1.01	0.05	1.02	0.06
C/BC/B-4-1200	14	1.01	0.05	1.02	0.06	1.02	0.05	1.02	0.06
C/BC/B-5-1200	14	1.02	0.06	1.02	0.06	1.02	0.06	1.02	0.06
All FE (each grade)		1.01	0.04	1.01	0.05	1.05	0.06	1.04	0.06
Collated Exp	34	mean	COV	mean	COV	mean	COV	mean	COV
		1.02	0.06	1.05	0.06				
All (FE and Exp)	202	mean	COV	mean	COV	mean	COV	mean	COV
		1.01	0.04	1.05	0.07				

344 **6.2 Direct Strength Method (DSM)**

345 The DSM can be considered as a generalised slenderness method for structural design and  
 346 was originally developed for cold formed steel sections (Schafer, 2008). The method aims to  
 347 overcome the complexity and limitations of the effective width method and combines the  
 348 interaction of local, flexural and distortional buckling of thin-walled members. The latter is not  
 349 discussed in this paper, as it is not relevant for the presently studied circular hollow sections.

350 Research has been carried out to extend the method's application to aluminium alloy  
 351 sections by Zhu and Young (2006b, 2009) and the applicability of the expressions proposed in  
 352 the aforementioned studies is examined hereafter. According to them, the axial strength of the  
 353 member ( $N_{DSM}$ ) is calculated as the minimum between the overall flexural buckling ( $N_{ne}$ ) and  
 354 the local buckling ( $N_{nl}$ ) strength. Equations (8), (9) are recommended to estimate these  
 355 strengths, where  $N_y$  is the yield load and is equal to  $A\sigma_{0.2}$ ,  $\lambda_1 = \sqrt{N_{ne}/N_{cr1}}$ , with  $N_{cr1}$  being the  
 356 critical elastic local buckling load,  $\lambda_c = \sqrt{N_y/N_{cre}}$ , where  $N_{cre}$  is the critical elastic flexural  
 357 buckling load. In a similar way, the bending strength of a member is the minimum between the  
 358 nominal flexural strength for lateral-torsional buckling ( $M_{ne}$ ) and the nominal flexural strength  
 359 for local buckling ( $M_{nl}$ ), which are calculated from Equations (11) and (12), respectively. In  
 360 these equations,  $M_y$  is equal to  $\sigma_{0.2}S_g$ , where  $S_g$  is the elastic cross-sectional modulus (i.e.,  
 361 referring to the extreme fibre in first yield), whilst  $\lambda_1 = \sqrt{M_{ne}/M_{cr1}}$ , with  $M_{cr1}$  being the critical  
 362 elastic local buckling moment. Upon calculation of these strengths, a linear interaction formula,  
 363 as shown in Equation (13) is recommended for members subjected to combined axial  
 364 compression and bending moment. Note that for the evaluation of  $N_{cr1}$  and  $M_{cr1}$ , the elastic local  
 365 buckling stress ( $f_{cr1}$ ) provided in Equation (14) has been used, both for the case of compression  
 366 and of bending (Buchanan et al., 2019), where  $\nu$  the material's Poisson's ratio.

$$N_{DSM} = \min(N_{ne}, N_{nl}) \quad (7)$$

$$N_{ne} = \begin{cases} (0.658\lambda_c^2) N_y & \text{for } \lambda_c \leq 1.5 \\ (0.877/\lambda_c^2) N_y & \text{for } \lambda_c > 1.5 \end{cases} \quad (8)$$

$$N_{nl} = \begin{cases} N_{ne} & \text{for } \lambda_1 \leq 0.713 \\ \left[1 - 0.15 \left(\frac{N_{cr1}}{N_{ne}}\right)^{0.3}\right] \left(\frac{N_{cr1}}{N_{ne}}\right)^{0.3} N_{ne} & \text{for } \lambda_1 > 0.713 \end{cases} \quad (9)$$

$$M_{DSM} = \min(M_{ne}, M_{nl}) \quad (10)$$

$$M_{ne} = \begin{cases} M_{cre} & \text{for } M_{cre} < 0.56M_y \\ \left(\frac{10}{9}\right) M_y \left(1 - \frac{10M_y}{36M_{cre}}\right) & \text{for } 0.56M_y \leq M_{cre} \leq 2.78M_y \\ M_y & \text{for } M_{cre} > 2.78M_y \end{cases} \quad (11)$$

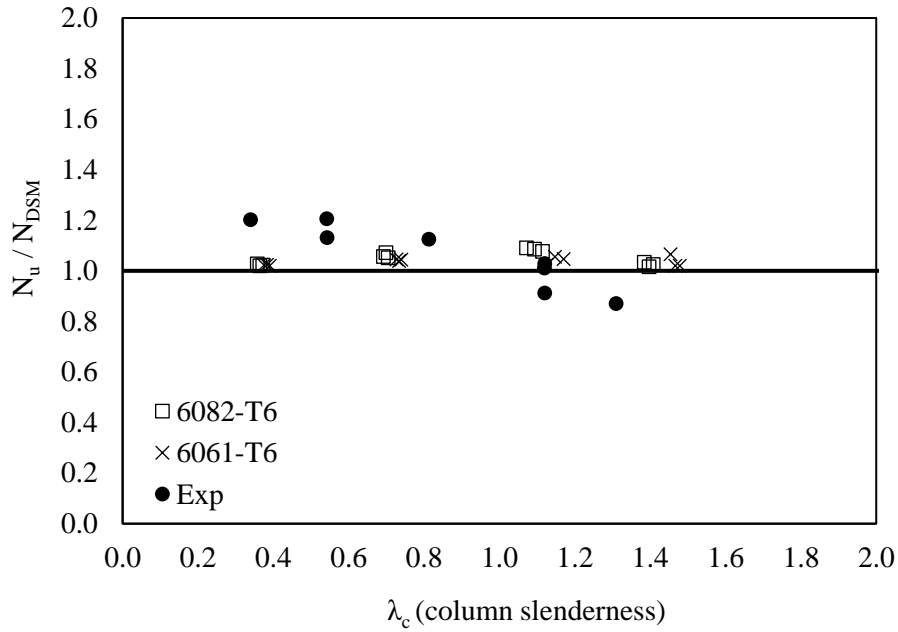
$$M_{nl} = \begin{cases} M_{ne} & \text{for } \lambda_l \leq 0.713 \\ \left[1 - 0.15 \left(\frac{M_{crl}}{M_{ne}}\right)^{0.3}\right] \left(\frac{M_{crl}}{M_{ne}}\right)^{0.3} M_{ne} & \text{for } \lambda_l > 0.713 \end{cases} \quad (12)$$

$$\left(\frac{N_{u,FE}}{N_{DSM}}\right) + \left(\frac{M_{u,FE}}{M_{DSM}}\right) \leq 1.00 \quad (13)$$

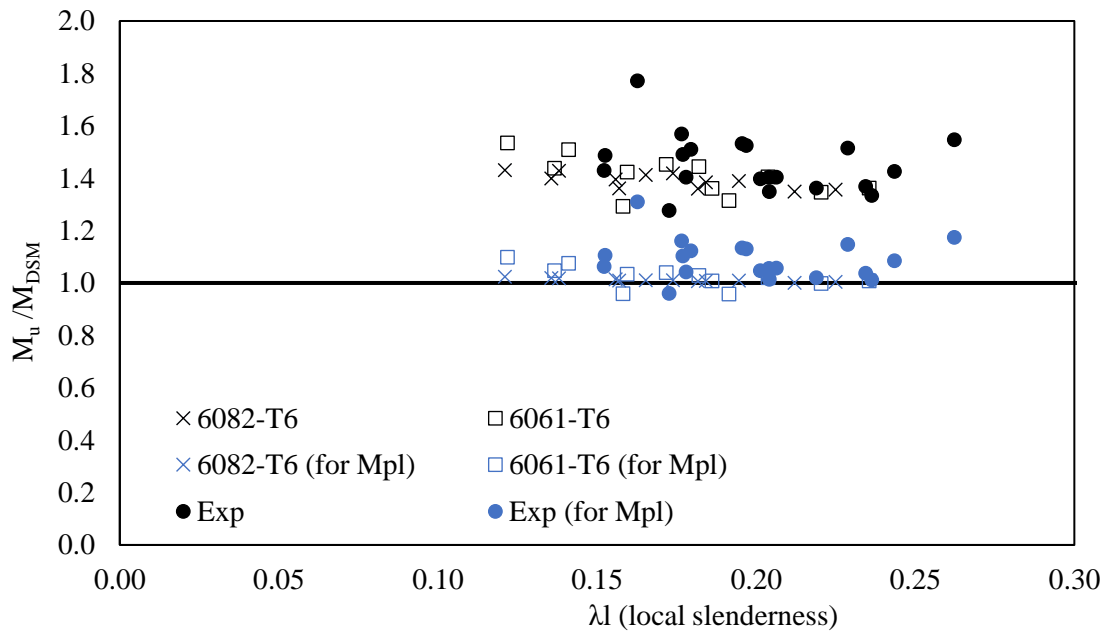
$$f_{crl} = \frac{E}{\sqrt{3(1-\nu^2)}} \frac{2t}{D} \quad (14)$$

367 The numerically obtained strengths are used to evaluate the accuracy of the DSM for  
 368 aluminium alloy circular hollow section members. Note that for the members investigated  
 369 herein  $\lambda_c \leq 1.5$  ( $0.36 \leq \lambda_c \leq 1.48$ ) as the study did not focus on very slender members, whilst  
 370 due to the low value of local slenderness,  $\lambda_l \leq 0.713$  ( $0.12 \leq \lambda_l \leq 0.24$ ), local buckling was  
 371 not a primary failure mode. In addition, distortional buckling does not occur in CHS sections  
 372 and lateral-torsional buckling was not an observed failure mode.

373 Firstly, the data for the cases of pure compression and of pure bending are utilised to  
 374 assess the predicted strengths for the two extreme cases. This is presented in Figure 13, where  
 375 the ultimate loads ( $N_u$ ) are normalised with the predicted strengths calculated using Equations  
 376 (7)-(9) ( $N_{DSM}$ ) and plotted against the column slenderness. It can be seen that  $N_{DSM}$  can  
 377 generally predict well the buckling strength. On the other hand, the method which was initially  
 378 formulated for thin-walled structures, does not consider any plastic bending moment, even for  
 379 sections with low cross-section slenderness values, as those studied herein. Hence, as shown  
 380 in Figure 13(b), the flexural strength of stocky CHS aluminium sections is clearly  
 381 underestimated by the DSM equation (i.e.,  $M_u/M_{DSM}$  well over unity). In the same figure, the  
 382 moment capacities have also been normalised with the plastic moment resistance ( $M_{pl}$ ), which  
 383 is evaluated multiplying the plastic section modulus with the material's proof strength, and as  
 384 can be observed, this leads to accurate flexural strength predictions.



(a) Buckling strength, based on 24 FE data of columns and 8 experimental reported column data



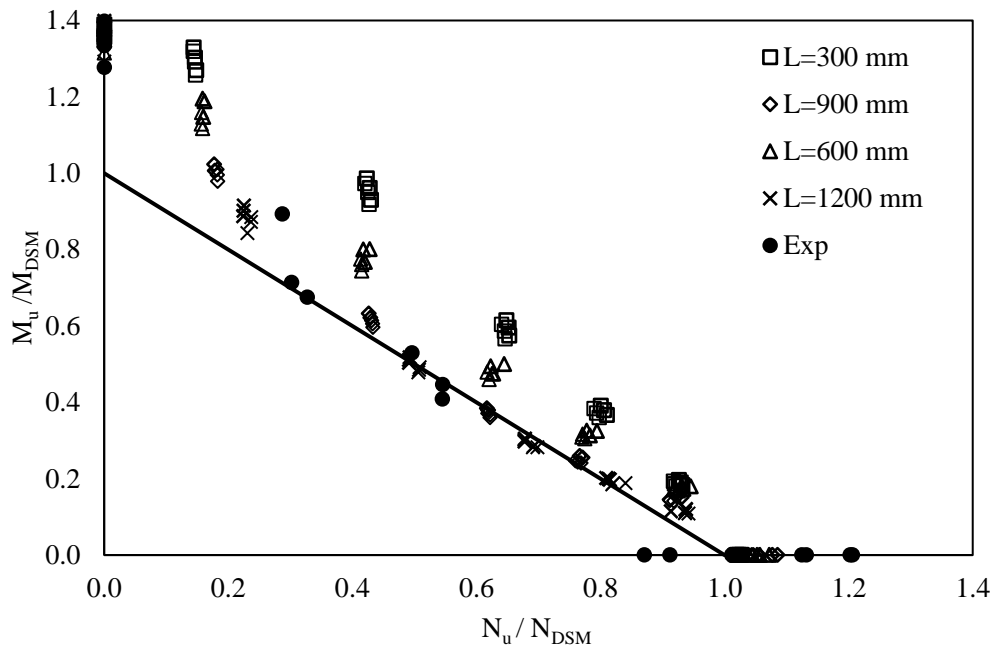
(b) Flexural strength, based on 24 FE data of beams and 20 experimental reported beam data

385 Figure 13: Assessment of DSM predicted strengths for the cases of pure compression and  
 386 pure bending.

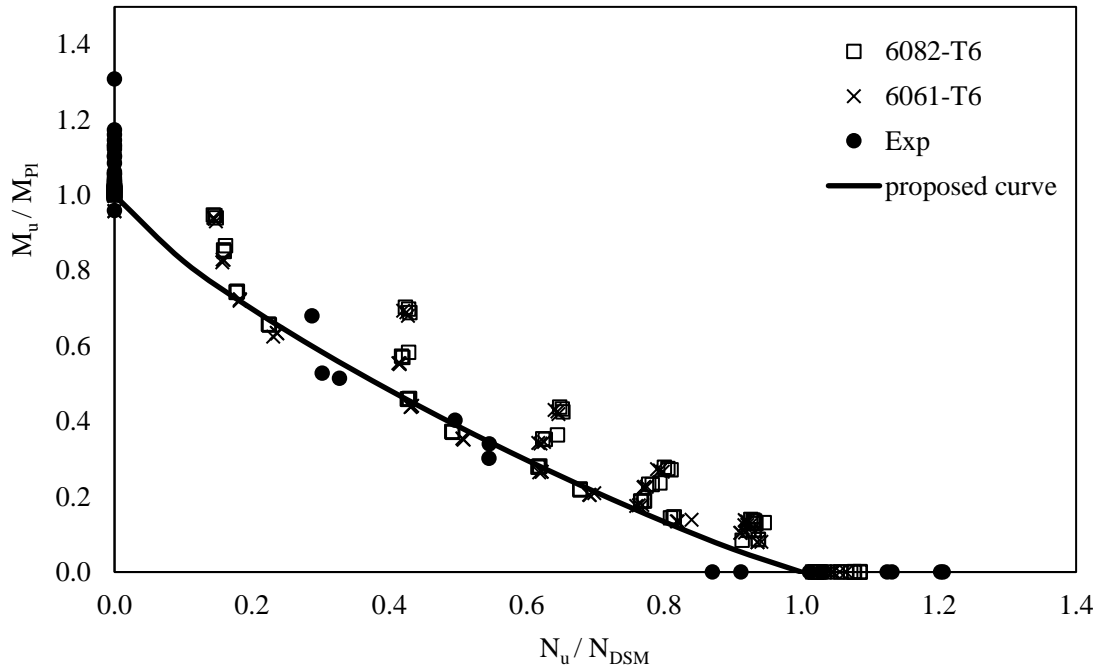
387 To evaluate the DSM recommendation for beam-columns (i.e., Equation (13)), Figure  
 388 14(a) presents the ultimate strengths normalised with DSM predictions and presented

389 together with the linear interaction curve. The results are shown with different markers  
 390 according to their length. As can be seen, for members subjected to a combination with  
 391 higher bending moment, the prediction is less accurate, as the normalised points diverge  
 392 from the interaction curve. This is mainly related to the conservative  $M_{ne}$  prediction, which  
 393 does not consider the plastic moment resistance achieved by stocky sections. To eliminate  
 394 this conservatism, the plastic moment resistance should be used in place of the elastic one.  
 395 Figure 14(b) presents the data normalised with the DSM buckling strength for the N  
 396 capacities ( $N_u/N_{DSM}$ ) and with the plastic resistance for the M capacities ( $M_u/M_{pl}$ ). On the  
 397 basis of normalised data, the interaction curve shown in Figure 14 and expressed in  
 398 Equation (15), is recommended in order to capture the response of aluminium alloy CHS  
 399 beam-columns with stocky cross-sections ( $\lambda_l \leq 0.30$ ).

$$\left[ \left( \frac{N_u}{N_{DSM}} \right)^{0.8} + \left( \frac{M_u}{M_{pl}} \right) \right] \leq 1.00 \quad (15)$$



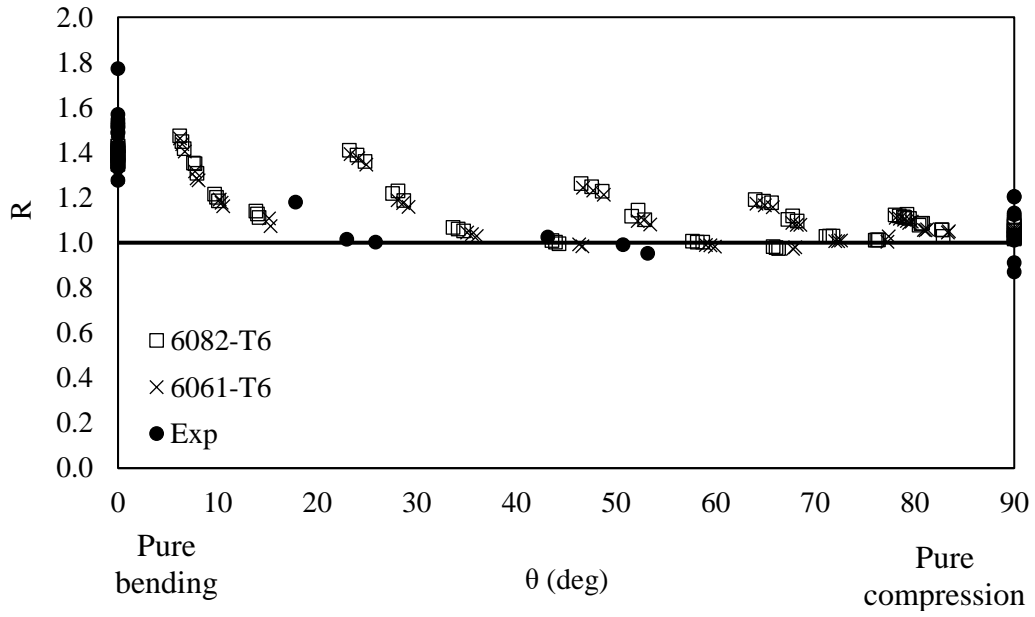
(a) Ultimate performance normalised with DSM strength - Design interaction curve based on DSM.



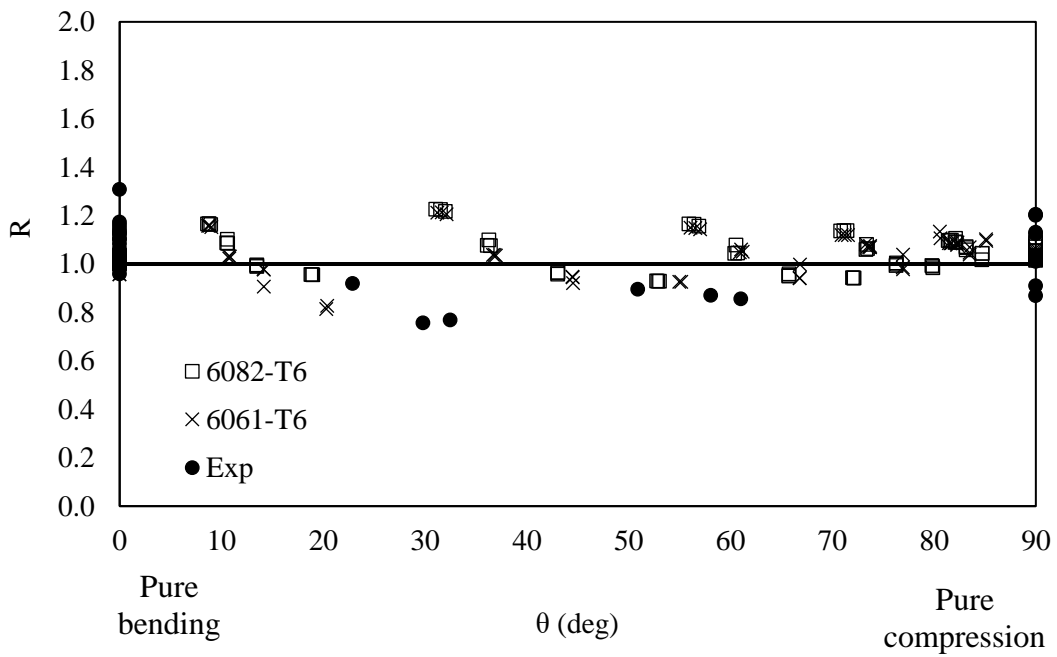
(b) Ultimate performance normalised with  $N_{DSM}$  and  $M_{PI}$  - Proposed interaction curve

400 Figure 14: Ultimate performance normalised with DSM strength for the two numerically  
 401 studied materials.

402 The linear interaction curve and the proposed interaction curve for stocky cross-  
 403 sections, are further assessed in Figure 15 and Table 6. As shown in Figure 15(a), the DSM  
 404 predictions appear less accurate and more scattered, compared to Figure 15(b). Similar  
 405 conclusions are drawn from Table 6, where the predicted capacities ratios show similar  
 406 performance for the two aluminium alloys. A mean value of  $R$  equal to 1.16 has been  
 407 achieved for the two studied materials for the DSM. The proposed modification for stocky  
 408 cross-sections significantly improves the  $R$  value to 1.04. In addition, the scatter reduces  
 409 and thus the COV which was relatively high for DSM (COV=0.15) improves using the  
 410 proposed modification (COV=0.07).



(a) Based on DSM strengths



(b) Based on  $N_{DSM}$ ,  $M_{pl}$  and proposed interaction curve

411 Figure 15: Assessment of DSM and of the proposed modification for stocky cross-  
 412 sections; results presented degree  $\theta$  of compression/bending.

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414

415 Table 6: Assessment of DSM and of the proposed modification for stocky cross-sections

416

Accuracy R; for the two studied materials.

Specimen	No FE	DSM				Proposal for stocky cross-sections			
		6082-T6		6061-T6		6082-T6		6061-T6	
		mean	COV	mean	COV	mean	COV	mean	COV
C/BC/B-3-300	14	1.24	0.12	1.23	0.12	1.12	0.07	1.11	0.06
C/BC/B-4-300	14	1.26	0.13	1.25	0.13	1.12	0.07	1.11	0.06
C/BC/B-5-300	14	1.27	0.13	1.26	0.14	1.12	0.07	1.11	0.06
C/BC/B-3-600	14	1.17	0.10	1.15	0.10	1.06	0.03	1.04	0.02
C/BC/B-4-600	14	1.20	0.10	1.15	0.09	1.08	0.03	1.04	0.04
C/BC/B-5-600	14	1.20	0.12	1.18	0.13	1.06	0.03	1.05	0.02
C/BC/B-3-900	14	1.12	0.11	1.09	0.12	1.01	0.05	0.99	0.04
C/BC/B-4-900	14	1.12	0.12	1.11	0.14	1.01	0.05	1.00	0.05
C/BC/B-5-900	14	1.13	0.13	1.12	0.16	1.01	0.05	1.00	0.06
C/BC/B-3-1200	14	1.07	0.12	1.07	0.10	0.98	0.04	0.98	0.06
C/BC/B-4-1200	14	1.08	0.14	1.08	0.15	0.99	0.04	0.99	0.05
C/BC/B-5-1200	14	1.09	0.14	1.10	0.18	0.99	0.05	0.99	0.06
All FE (each grade)		1.16	0.13	1.15	0.13	1.04	0.07	1.03	0.07
Collated Exp	34		mean 1.29	COV 0.18			mean 1.04	COV 0.12	
All (FE and Exp)	168		mean 1.18	COV 0.15			mean 1.04	COV 0.07	

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### 6.3 Aluminium Design Manual

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The design recommendations of the Aluminium Design Manual (Aluminum design manual, 2015) for the aluminium alloy members are assessed in this section. The design strengths for axial compression ( $N_{AA}$ ) are defined as the minimum among the states of member buckling  $N_{cm}$ , local buckling  $N_{cl}$  and interaction between member buckling and local buckling  $N_{ce}$ .  $N_{cm}$  is given by Equation (17), where  $F_c$  is the compressive yield strength,  $B_c$  and  $C_c$  are the member buckling constant intercepts, and  $D_c$  is the member buckling constant slope, as defined in AA (2015). The bending strength is the minimum between the yielding, rupture ( $M_{np,u}$ ), local buckling ( $M_{nlb}$ ), and the lateral-torsional buckling ( $M_{nmb}$ ). The yield bending moment was the failure moment for the sections examined in this study. Formulae for their evaluation of all cases are provided in AA (2015). When a member is subjected to combined axial compression and bending among two axes ( $y$  and  $z$ ), the linear interaction formula given in Equation (19) is recommended.

431 Only one bending axis was considered in this numerical investigation and hence the  
 432 bending moment on the other axis is zero.

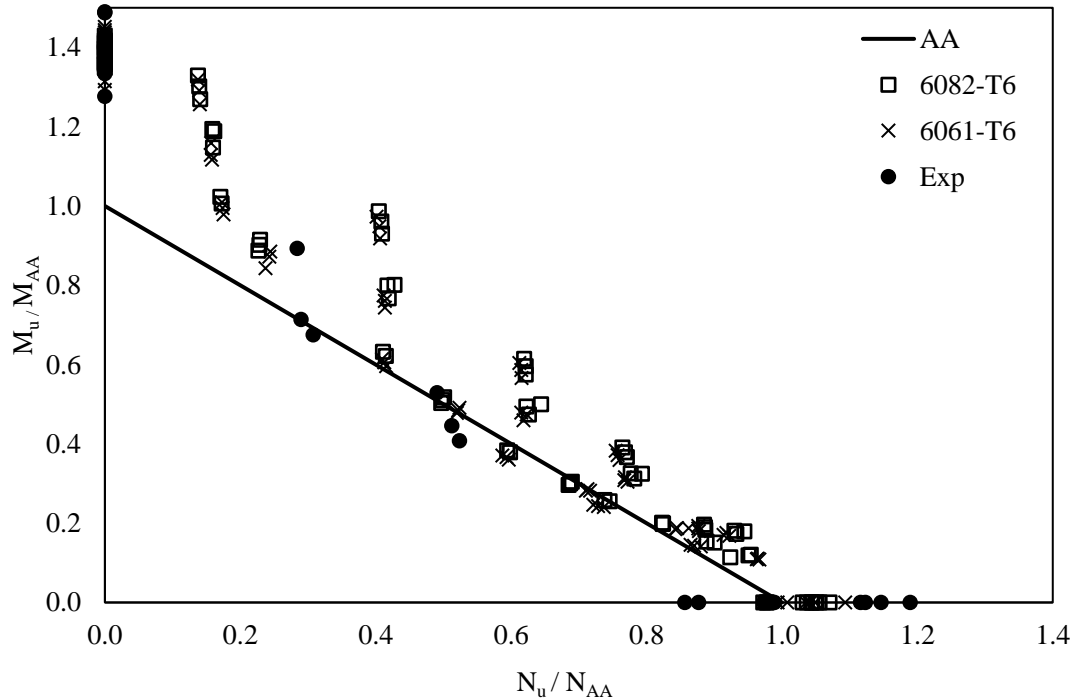
$$N_{AA} = \min(N_{cm}, N_{cl}, N_{ce}); \quad (16)$$

$$N_{cm} = F_c A_g = A_g \begin{cases} F_{cy} & \text{for } \lambda \leq \lambda_1 = (B_c - F_{cy})/D_c \\ B_c - D_c \lambda (0.85 + 0.15(C_c - \lambda)/(C_c - \lambda_1)) & \text{for } \lambda_1 \leq \lambda \leq C_c \\ 0.85 \pi^2 E / \lambda^3 & \text{for } \lambda \leq C_c \end{cases} \quad (17)$$

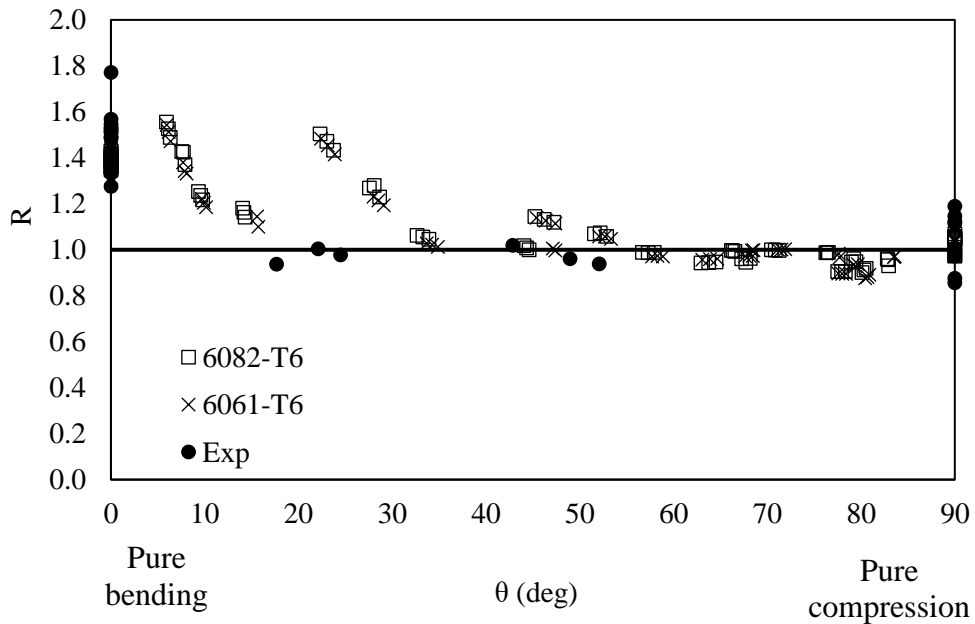
$$M_{AA} = \min(M_{np,u}, M_{nlb}, M_{nmb}) \quad (18)$$

$$\left| \left( \frac{N_{u,FE}}{N_{AA}} \right) + \left( \frac{M_{u,FE}}{M_{AA,y}} \right) + \left( \frac{M_{u,FE}}{M_{AA,z}} \right) \right| \leq 1.00 \quad (19)$$

433 The accuracy of AA (2015) is assessed in Figure 16, Figure 17 and in Table 7. Similar  
 434 to the DSM and due to the inaccurate prediction of the bending moment, the interaction  
 435 design strength appears quite conservative, particularly for members primarily under  
 436 bending moment. The method leads to R with a mean value of 1.14, with COV equals to  
 437 0.18 for both aluminium alloys. This observation is in line with the outcomes reported for  
 438 rectangular hollow section aluminium alloy beam-columns in Hu et al. (2020).



439  
 440 Figure 16: Ultimate performance normalised with AA strength for the two numerically  
 441 studied materials (Design curve based on AA).



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444 Figure 17: Assessment of AA; results presented degree  $\theta$  of compression/bending.

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Table 7: Assessment of AA - Accuracy R; for the two studied materials.

		6082-T6		6061-T6	
		mean	COV	mean	COV
C/BC/B-3-300	14	1.18	0.21	1.17	0.21
C/BC/B -4-300	14	1.19	0.22	1.19	0.22
C/BC/B -5-300	14	1.21	0.23	1.21	0.23
C/BC/B -3-600	14	1.14	0.15	1.12	0.15
C/BC/B -4-600	14	1.16	0.17	1.12	0.15
C/BC/B -5-600	14	1.16	0.18	1.15	0.18
C/BC/B -3-900	14	1.08	0.14	1.06	0.15
C/BC/B -4-900	14	1.09	0.15	1.07	0.17
C/BC/B -5-900	14	1.10	0.17	1.09	0.20
C/BC/B-3-1200	14	1.07	0.14	1.06	0.11
C/BC/B-4-1200	14	1.08	0.14	1.08	0.15
C/BC/B-5-1200	14	1.09	0.15	1.10	0.18
All FE (each grade)		1.13	0.17	1.12	0.17
Collated Exp	34		1.27	1.29	
All (both grades)	202		mean	COV	
			1.14	0.18	

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448 **6.4 Reliability analysis**

449 In order to assess the reliability of the proposed design method, a statistical analysis was  
 450 conducted. Initially, a comparison between the predicted capacities ( $R_{pred}$ ) against the  
 451 experimental/FE obtained capacities ( $R_{exp/FE}$ ) is presented in Figure 18. As can be seen, the  
 452 points are generally below the diagonal unity line, showing predictions on the safe side.  
 453 Following the provisions of annex D EN 1990 (CEN, 2002), the proposed method was then  
 454 statistically validated. Table 8 summarises the following key statistical parameters: the number  
 455 of tests and FE simulations ( $n$ ), the design (ultimate limit state), the fractile factor ( $k_{d,n}$ ), the  
 456 average ratio of test (or FE) to model resistance based on a least-squares fit to all the data ( $\bar{b}$ ),  
 457 the CoV of the tests and FE simulations relative to the resistance model ( $V_{\delta}$ ), the combined  
 458 CoV incorporating both model and basic variable uncertainties ( $V_r$ ) and the partial safety factor  
 459 for member resistance ( $\gamma_{M1}$ ). Based on the reliability analysis considerations provided in Su et  
 460 al. (2017), the material over-strength of HSS was taken equal to 1.10 with a CoV of 0.050,  
 461 while the CoV of geometric properties was assumed to be equal to 0.02 (Gkantou et al., 2017b).  
 462 The variation between the experimental and the numerical results was also considered  
 463 ( $V_{FEM}=0.1$ ). Performing a first-order reliability method in accordance with the Eurocode target  
 464 reliability requirements, the partial factors were evaluated. The derived partial safety factors  
 465 for all assessed methods are presented in Table 8. The highest safety factor is required for the  
 466 AA method, while for the proposed method  $\gamma_{M1}$  was found to be  $\gamma_{M1}= 1.04$ , which is lower  
 467 than the partial safety factor currently employed to Eurocode 9 which is equal to 1.10.

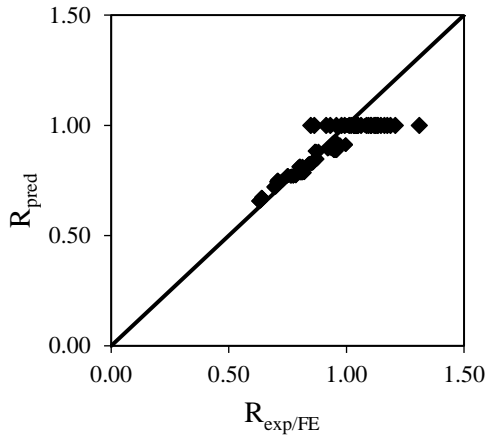
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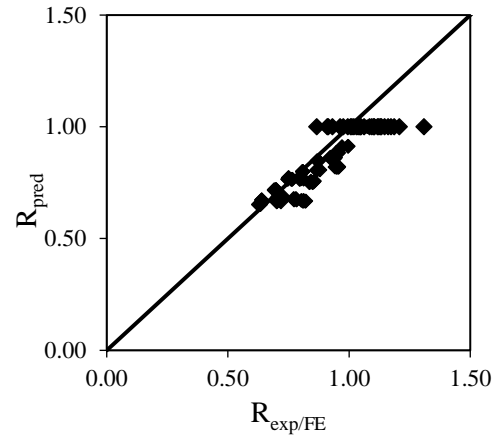
Table 8: Reliability analysis parameters

<b>Design methods</b>	<b>n</b>	<b><math>k_{d,n}</math></b>	<b><math>\bar{b}</math></b>	<b><math>V_b</math></b>	<b><math>V_r</math></b>	<b><math>\gamma_{m1}</math></b>
<b>EC9 - Method A</b>	202	3.139	1.031	0.057	0.100	1.01
<b>EC9 - Method B</b>	202	3.139	1.052	0.066	0.105	1.02
<b>DSM</b>	202	3.139	1.211	0.146	0.168	1.13
<b>DSM - proposal</b>	202	3.139	1.044	0.084	0.117	1.04
<b>AA</b>	202	3.139	1.180	0.177	0.195	1.19

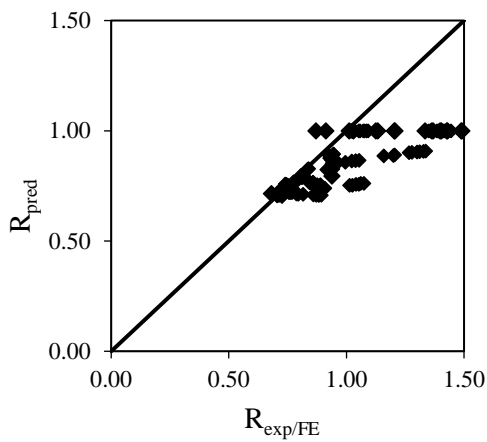
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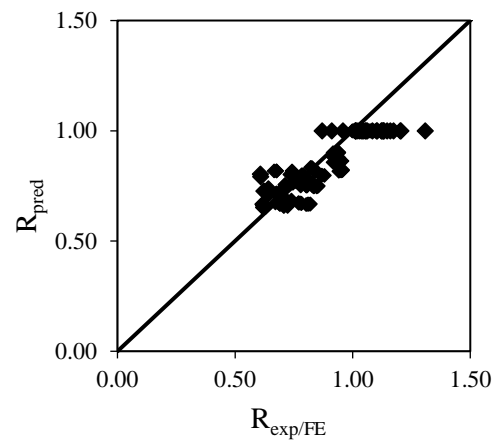
(a) Eurocode – Method A ( $\gamma_{m1} = 1.01$ )



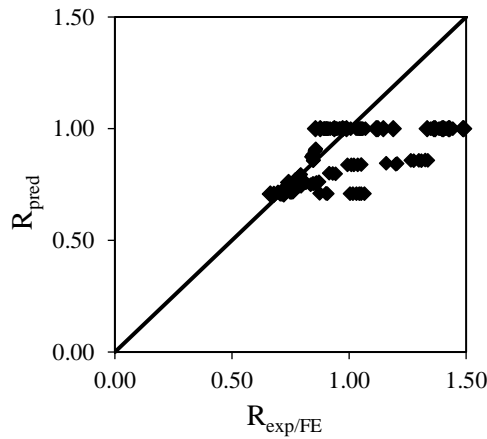
(b) Eurocode – Method B ( $\gamma_{m1} = 1.02$ )



(c) DSM ( $\gamma_{m1} = 1.13$ )



(d) DSM proposed ( $\gamma_{m1} = 1.04$ )



AA ( $\gamma_{m1} = 1.19$ )

471

472 Figure 18: Comparison between predicted ( $R_{pred}$ ) and experimental/numerical capacities

473

( $R_{exp/FE}$ ) for the assessed methods.

## 474 7. Conclusions

475 This paper presented a comparative assessment of design models available for the design  
476 of aluminium alloy CHS members with stocky sections subjected to compression and bending.  
477 In order to ensure an adequate validation of the FE model, a total of 34 experiments (20 beams,  
478 8 columns and 6 beam-columns) based on three research papers were used. A comprehensive  
479 numerical investigation on members subjected to a range of eccentric loading conditions was  
480 presented. The results were presented graphically on the M-N interaction curves, whilst the  
481 predicted capacity ratios were also plotted against the angle  $\theta$  and tabulated. Subsequently, the  
482 numerical results combined with the test data were used for the assessment of the European  
483 design provisions, the Direct Strength Method and the Aluminium Design Manual. The main  
484 observations and conclusions are listed below.

- 485 (1) A mesh convergence and geometric imperfection sensitivity study were performed,  
486 demonstrating that  $D/20$  was the optimum mesh size for all members and that the  
487 imperfection amplitude  $t/10 + L/1000$  obtained the best agreement between the  
488 experimental and numerical results.
- 489 (2) The highest accuracy was achieved by the European design provisions, and in particular  
490 for the case of  $\psi_c = 1.3\chi$  for Class 1 and Class 2 cross-sections. This method resulted in  
491 a mean value for R equal to 1.01.
- 492 (3) The most conservative predictions were given by the DSM method, resulting in R with  
493 mean values of 1.16 and 1.15, based on the FE data of 6082-T6 and 6061-T6,  
494 respectively.
- 495 (4) Taking into account the DSM buckling strength and the plastic moment resistance, a  
496 new interaction curve is proposed that can accurately predict the behaviour of members  
497 made of stocky circular hollow sections and subjected to combined compression and  
498 bending (mean R value of 1.04).
- 499 (5) The design method with the lowest design consistency was AA, resulting in COV equal  
500 to 18%.
- 501 (6) The design proposal presented herein is based on the results of CHS members made  
502 from 6082-T6 and 6061-T6. It is expected that any aluminium alloy material with  
503 similar material properties, in terms of overall response, hardening and ductility  
504 parameters, could be applicable to this proposal. However, further research on different  
505 structural aluminium alloys would be recommended to support this assumption.

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